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NAVORD REPORT

4259

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HEAT - TRANSFER CHARACTERISTICS OF A HEMISPHERE CYLINDER
AT HYPERSONIC MACH NUMBERS

FC

11 APRIL 1957



U. S. NAVAL ORDNANCE LABORATORY
WHITE OAK, MARYLAND

Aeroballistic Research Report 336

HEAT-TRANSFER CHARACTERISTICS OF A HEMISPHERE
CYLINDER AT HYPERSONIC MACH NUMBERS

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ABSTRACT: The heat-transfer characteristics of the laminar compressible boundary layer on a hemisphere cylinder have been investigated at free-stream Mach numbers of 5, 6.5, and 8. The Reynolds number based on free-stream conditions and model diameter was varied from 70,000 to 700,000. Various conditions of steady-state heat transfer to the model were realized by circulating a coolant through the model, and by varying the tunnel supply air temperature. The wall to stagnation temperature ratio was varied from 0.43 to 0.75. Optical observations and Pitot pressure surveys of the boundary layer showed it to be laminar on both the hemisphere and the cylindrical afterbody. The heat transfer was evaluated from the temperature differences measured across the model wall under steady-state conditions. Over the hemisphere, the local non-dimensional heat-transfer parameters are, on the average, approximately twenty percent larger than predicted for an isothermal body by Korobkin's modified incompressible theory.

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This report is an account of the hemisphere cylinder heat-transfer program carried out in the NOL 12 x 12 cm Hypersonic Tunnel No. 4.

Knowledge of the heat-transfer characteristics of blunt-nosed bodies has become of particular interest. The blunt nose alleviates some of the design difficulties resulting from the high rates of heat transfer and low heat capacity near the nose of pointed bodies.

A portion of the results contained in this NAVORD Report was presented at the 24th Annual Meeting of the Institute of Aeronautical Sciences in January 1956. The present report contains additional results and a more detailed analysis of the data as well as a complete tabulation of the experimental results.

This work was jointly sponsored by the U. S. Naval Bureau of Ordnance and the U. S. Air Force. It was carried out under Tasks NOL-133-1-56, and NOL-291.

The authors wish to express their indebtedness to Drs. R. E. Wilson, R. K. Lobb and Mr. I. Korobkin for many stimulating discussions during the course of the investigations. A large portion of the numerical evaluation was done by Mr. Moon H Cha. Mr. R. Garren, Jr., in addition to participating in the tests, was largely responsible for the model design and test preparations.

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SYMBOLS

C_p	pressure coefficient
D	model diameter
h	heat-transfer coefficient
k	thermal conductivity of air
k_m	thermal conductivity of model material
M	Mach number
Nu	Nusselt number
p	static pressure
Pr	Prandtl number
Re	Reynolds number
Δr	wall thickness of model
T	temperature
ΔT	temperature difference across wall
T_0	stagnation temperature
T_{eff}	effective temperature (surface temperature for zero heat transfer)
u	velocity
x	model contour length, measured from stagnation point
Δy	effective model wall thickness, defined by Equations 3 and 4
α	angular position, measured from stagnation point
β	velocity gradient at stagnation point
μ	absolute viscosity of air
ν	kinematic viscosity
ρ	density of air

Subscripts

- D diameter used as characteristic length
- e adiabatic wall conditions
- L local conditions at outer edge of model boundary layer
- w conditions at or on model wall
- x contour length used as characteristic length
- ∞ free-stream conditions at infinity

HEAT-TRANSFER CHARACTERISTICS OF A HEMISPHERE CYLINDER AT HYPERSONIC MACH NUMBERS

INTRODUCTION

1. When the present investigations were initiated, experimental information on the heat-transfer characteristics of laminar compressible boundary layers on blunt-nosed bodies were still rather limited inspite of the great practical interest in such body shapes. Most of the available data were obtained at Mach numbers below 5, and for small rates of heat transfer (references a-c). In each of these investigations attempts were made to simulate isothermal bodies. For these data, incompressible and modified compressible theories represent the over-all trend.

2. In order to realize larger rates of heat transfer, and to perform the investigations at hypersonic Mach numbers, the present studies were carried out on a cooled hemisphere cylinder in the NOL Hypersonic Tunnel No. 4 (reference d), at Mach numbers 5, 6.5, and 8. Free-stream Reynolds numbers based on model diameter ranged from 7×10^4 to 7.7×10^5 , and the model wall to stagnation temperature ratios from 0.43 to 0.75.

Description of Models and Experimental Procedure

3. The pressure and heat-transfer models are 3.8 cm diameter hemisphere cylinders made of type 302 stainless steel. They have a constant wall thickness, and can be cooled or heated internally to temperatures ranging from 210°K to 800°K. Pressure data were obtained at 10 positions along the hemispherical nose and the cylindrical afterbody with the pressure model shown in Figure 1. The pressure orifices, arranged in a spiral starting at the stagnation point, are 0.635 mm in diameter, which corresponds to an angle of 1.9 degrees on the model, or to an arc length, $\Delta x/D$, of 0.0167. Mercury manometers or oil manometers which have reading accuracies of ± 0.1 mm Hg and ± 0.001 mm Hg, respectively, were used to measure the pressures. A reference pressure orifice is located in both the pressure and the heat-transfer model at $x/D = 0.869$.

4. The heat-transfer model, shown partially assembled in Figure 2, has thermocouples located at 11 stations on both

the exterior and interior wall.* Type 302 stainless steel was selected for the models because its thermal conductivity varies linearly with temperature over the entire range considered for the tests. Also, the coefficient of thermal conductivity of this material is sufficiently small to assure a fair accuracy in measuring temperature differences across the model wall.

5. The coolant (silicon oil DC 200, 2 centistoke) enters the model through a tube which is concentric to the cylinder, and is then discharged into the hemisphere nose. The coolant returns through the annular clearance between the inlet tube and the model wall. The main entrance and exit passages of the coolant are part of the model support. The cooling system requirements were calculated by equaling the heat transfer from the air to the model to that from the model to the coolant. This is done by assuming the following: (1) the major contribution to the over-all heat transfer is due to the heat transfer to the hemispherical nose of the model; (2) the distribution of the latter can be predicted by the theory of reference (a); and (3) the absolute value at the stagnation point can be taken as the mean value of the experimental data reported in reference (a). From this procedure, together with the available engineering data for the silicon oil, a circulation rate of 4 gallons/minute was computed if the bulk temperature rise of the coolant should remain smaller or equal to 1°C for all test conditions. The measured temperature rise for this flow rate was well within the predicted limit, except at the lowest coolant temperature (about 210°K) where it amounted to 3°C . An exterior, thermostatically controlled temperature bath maintained the coolant at any desired temperature within the range from 210°K to 370°K . For higher temperatures, hot air was circulated through the model at a rate that its temperature rise or drop between entrance and exit was negligible.

6. The interior thermocouples are made of 30-gauge iron

*A first model provided for 3 thermocouples at each of the 11 stations. The model wall thickness was calculated, using the results of reference(a), to vary from the stagnation point to the shoulder in such a fashion that for a constant interior temperature also a constant exterior temperature should have been obtained. The thermocouples were then placed on calculated isothermes in the model wall. Preliminary test results did not verify that the thermocouples were located on isothermes.

The model described above has a simpler construction which alleviated some of the difficulties in the assembly but has the disadvantage of providing for only 2 readings at each station.

constantan Ceramo wires.* The exterior thermocouples, located radially above each interior one, are made of 36-gauge iron constantan wires and are imbedded in grooves on the model surface with an insulating cement.** The sizes of the grooves, of the thermocouple junctions, and of the spherical recesses into which the interior junctions are welded, are such that the thermocouples are located with an accuracy better than ± 0.025 cm. To reduce the conduction losses along the exterior thermocouple wires, the grooves into which the wires are imbedded form at least a semi-circle around the model and then they lead the wires straight back to the base of the after-body. The exterior and interior temperatures were recorded on two synchronized operating 12-point Brown recorders which have printing intervals of two seconds. Temperature readings, accurate to $\pm 0.1^{\circ}\text{C}$, were taken after practically steady-state conditions were reached, which required 5 to 10 minutes operation at the desired test conditions.

7. In addition to the pressure and temperature distributions, measurements were also made of the surface temperatures for zero heat transfer. For these tests hot air was circulated through the model and the temperature adjusted until a locally zero temperature gradient was observed successively for each station.

8. Information on the flow pattern and the condition of the boundary layer around the model was obtained from schlieren observations and boundary-layer surveys.

Data Reduction

9. The free-stream conditions were determined from measurements of the wall static and Pitot pressures in the test section, and from recordings of the stagnation temperature, T_0 , in the nozzle inlet. The Rayleigh formula was used to compute the Mach number. The local flow conditions around the model, in terms of M_L , T_L , ρ_L , and u_L were calculated from the measured pressure distributions and T_0 . In making these calculations, the flow was assumed to expand isentropically over the model, and the stagnation point pressure was set equal to the Pitot pressure. The boundary-layer surveys indicated that these assumptions are justified.

*Manufactured by Thermo Electric Co., Inc., Fair Lawn, New Jersey.

**Technical B copper cement, manufactured by W. V-B Ames Co., Fremont, Ohio

10. Other relations and quantities used in the further evaluation of the data are the pressure coefficient

$$c_p = (p_L - p_\infty) / \left(\frac{1}{2} (\rho_\infty u_\infty^2) \right) \quad (1)$$

The Nusselt number based on either the model diameter or the arc length are:

$$Nu_D = \frac{hD}{k} = \frac{qD}{(T_e - T_w)k_{air}} ; \quad Nu_x = \frac{hx}{k} = \frac{qx}{(T_e - T_w)k_{air}} \quad (2)$$

The heat transferred from the air to the model, q , is calculated from the heat conducted through the model shell where the contribution of radiation heat transferred to or from the model is neglected. The heat transfer, q , is equal to the average conductivity, k_m , of model material multiplied by the temperature gradient at the model surface (reference 1). Assuming one-dimensional conduction, that is neglecting the transverse conduction along the body contour, the temperature gradient at the model surface is (reference 1),

$$\frac{\Delta T}{\Delta y} = \frac{\Delta T}{D \Delta r} \quad (D - 2 \Delta r) \quad (3)$$

for the hemisphere,

$$\frac{\Delta T}{\Delta y} = \frac{2 \Delta T}{D \log_e \frac{D}{D - 2 \Delta r}} \quad (4)$$

for the cylinder. The adiabatic wall temperature T_e for a laminar boundary layer is

$$T_e = (Pr)^{1/2} (T_\infty - T_L) + T_L \quad (5)$$

Consistent with the usual representations of heat-transfer data, the Nusselt numbers are ratioed to either $(\beta D^2 / \nu)^{1/2}$ or $Re_x^{1/2}$ where β is the velocity gradient at the stagnation point

$$\beta = \left(\frac{du_L}{dx} \right) \propto -0 \quad (6)$$

and

$$Re_x = \frac{u_L x}{\nu} \quad (7)$$

For the viscosity of air Sutherland's formula was used (NBS Table 2.39). The thermal conductivity of air was obtained using the empirical formula

$$k = \frac{0.6325 \times 10^{-5} T^{3/2}}{T + 245.4 \times 10^{-12}/T} \text{ cal/cm sec } ^\circ\text{C} \quad (8)$$

(NBS Table 2.42). Prandtl number values were taken from the NBS Table 2.44. The thermal conductivity of the 302 stainless steel was measured on a sample cut from the model stock by the National Bureau of Standards (reference f).

11. In addition to computing the heat-transfer coefficient on the basis of $(T_e - T_w)$, experimental values of the surface temperature for locally zero heat transfer, T_{eff} , have also been used instead of T_e . In general, the measured T_{eff} data are not free from radiation losses, since the tunnel walls are, in some cases, cooler than the model by a factor of 2.5. These data were therefore corrected assuming a cylindrical geometry of tunnel and model, and an emissivity of 0.7.

RESULTS

Pressure Measurements

12. The Mach number distributions obtained from the pressure measurements over the model are shown in Figure 3. The distribution over the hemisphere is consistent with the results of references (a-c). Over the cylindrical afterbody the Mach number continues to rise slowly approaching a value which appears to depend on the free-stream Mach number.

13. The measured pressure distributions presented as pressure coefficients are shown in Figure 4. The absolute values of the pressure coefficient for the forward half of the hemisphere are lower than Newtonian theory predicts, but are closely represented by a curve calculated using Pitot pressure to determine C_{pmax} . The agreement is good for small values of x/D , but the experimental C_p data deviate from the calculated $\cos^2 2 x/D$ curve at values of x/D larger than about 0.5. Near this point,

for the Mach number range 5 to 8, the pressure distribution has the same slope as one assuming a Prandtl-Meyer expansion and good agreement with the data is obtained if the Prandtl-Meyer calculation is started at the point of equal slopes.

14. The velocity gradient at the stagnation point used in correlating the heat-transfer data was determined from graphs of the velocity distribution over the hemisphere. The local velocity was calculated from the measured pressure distribution and the stagnation temperature. Values of the normalized velocity gradient, $\delta D/u_{\infty}$, are plotted in Figure 5. Values obtained from the experimental Mach number distributions are compared with the data of references (a, b, c, h, i, j, and k) and with a theoretical curve based on the Newtonian pressure distribution. The present velocity gradients are about 10 percent higher than theory predicts.

Boundary-Layer Surveys

15. The boundary layer was surveyed at 5 stations along the model at a free-stream Mach number, M_{∞} , of 8. For these tests, hot air was circulated through the model at the temperature necessary to achieve practically zero heat transfer at the stagnation point. Pitot probes of a half-height of 0.005 inch were used for the surveys. To evaluate the Mach number distribution across the boundary layer, the measured Pitot pressures were referred to the local wall-static pressure value. The surveys shown in Figure 6 are characteristic of laminar boundary layers. A slight overshoot was measured beyond the juncture of the hemisphere and the cylinder $x/D = 0.869$, which is probably due to an over-expansion at the juncture. For each station, the Mach number at the outer edge of the boundary layer agrees closely with the Mach number, M_L , obtained from the ratio of the local wall-static pressure and the pressure measured at the stagnation point. A comparison of these data with theory was felt not to be justified because their further evaluation would require measured distributions of the boundary-layer temperature which were not obtained during the present investigations.

16. For the free-stream Mach numbers 6.5 and 5, schlieren optical observations were made which also confirmed a laminar boundary layer for the entire model.

Heat-Transfer Measurements

17. The measured temperature distributions deviate considerably from an isothermal wall. For small rates of heat

transfer, the temperature varied from 330°K at the stagnation point to 304°K at the shoulder (see Appendix A $M_{\infty} = 5.11$). For higher rates of heat transfer a variation from 368°K to 266°K was observed (Appendix A $M_{\infty} = 4.9$). Because the inside wall temperature remained relatively constant in all cases, the non-isothermal surface temperature distributions are reflected in the relative temperature differences which are shown in Figure 7.

18. The calculation of surface heat transfer from the measured temperature differences was based on the assumption of one-dimensional heat flow across the model wall. The deviation from an isothermal temperature distribution shows this is only a first approximation. An attempt was made to obtain a better approximation by evaluating the effect of the conduction along the model. The principal procedure tried assumes no conductivity variation and uses the solution of Laplace's equation in spherical coordinates with flow axis symmetry with the inside and outside temperature distributions as the boundary conditions. This procedure did not give consistent results, mainly because the number of measurements were insufficient to define the derivatives of the distribution on which the longitudinal conduction depends.

19. The over-all behavior of the curves of Figure 7 differs at two places from the distribution predicted by incompressible theory. The temperature difference has a maximum at $x/D = 0.0975$ ($\alpha \sim 11^\circ$), and not at $\alpha = 0$. An increased temperature difference occurs, in some cases quite pronounced, at $x/D = 0.393$ ($\alpha = 45^\circ$) which corresponds approximately to the intersection of the sonic line with the body contour. Measurements in addition to those presented in Figure 7 showed that the maximum off the stagnation point decreases with decreasing rate of heat transfer and disappears for the case of zero heat transfer. Recently published data (reference h, Figure 10) obtained by the transient technique indicates the temperature distribution develops a maximum approximately 4 1/2 degrees from the stagnation point as steady-state conditions are approached.

20. The heat-transfer data, expressed as $Nu_x/(Re_x)^{1/2}$, are shown in Figures 8a through 8h. The numerical values for all 34 surveys are given in the tables. The tabulated values of $Nu_x/(Re_x)^{1/2}$ are based on T_w and T_e . For the graphical representations, the reference temperatures T_w and T_{eff} have been used. The conversion quantity $(T_e - T_w)/(T_{eff} - T_w)$ is included in the tables.

DISCUSSION OF DATA

21. In general, the plots of the non-dimensional heat-transfer

parameters exhibit, in either representation, the behavior already indicated in Figure 7. For each heat-transfer condition and Mach number, the individual distributions show a rather characteristic behavior which repeats itself, more or less, with variation in free-stream Reynolds number. Some characteristic features are, however, clearly shown by all distributions, the peaks at $x/D = 0.0975$ ($\alpha \sim 11^\circ$) and at $x/D = 0.393$ ($\alpha = 45^\circ$), as well as the rather large values of the parameter at the shoulder. It is difficult to draw any conclusions regarding a trend with either Mach number, Reynolds number or heat-transfer rate. The band representing all the data is considerably wider than can be explained by experimental scatter, which is of the order of ± 5 percent.

22. The effect of reference temperature (at which the properties of air are evaluated) is illustrated in Figure 9. While the selection of wall temperature or temperature at the outer edge of the model boundary layer as reference temperature has very little effect, not more than about 1.5 percent for the T_w and T_L values encountered, the use of T_{eff} instead of T_e has the tendency to flatten the distributions.

23. In Figure 10a and 10b are shown the present data, together with other experimental results, and with theory. The latter has been done only to orient the present data with reference to theoretical predictions which are based on the concept of an isothermal body and one-dimensional heat conduction. In comparison to Silbulkin's stagnation point value of 0.661 (reference g) all data are high. All experimental data exhibit the same general behavior over the front portion of the hemisphere. The stagnation point value is not a maximum value. (The Crawford and McCauley data, reference h, show a similar effect near the stagnation point as their test approaches steady-state conditions.) Like the present results, Stine and Wanlass' data show a peak value at $\alpha \sim 11^\circ$, and another one at $\alpha = 45^\circ$, and 60° , respectively. Beyond this point, Stine and Wanlass' data decrease rapidly and approach the flat plate value. Not so the present data, the behavior of which is consistent with Gruenewald and Fleming's and Korobkin's results. The numerical values are, on the average, 20 percent larger than predicted by Korobkin's modified incompressible theory.

24. Finally, in Figure 11, the local Stanton number values computed from the data obtained up to about 75° angular position are compared with $St = 0.763 \text{ Pr}^{0.6} \text{ Rex}^{-0.5}$ (reference a). Though the experimental data exhibit the same slope as the above relation, the numerical values are, on the average, about 12.5 percent larger.

CONCLUDING REMARKS

25. The heat-transfer characteristics of a hemisphere-cylinder have been investigated at hypersonic Mach numbers from 5 to $M = 8$, and model wall to stagnation temperature ratios from 0.43 to 0.75. The data have been evaluated from temperature difference measurements made across the wall on a cooled model after practically steady-state conditions were reached.
26. The distributions of the pressure coefficient ratio $C_p/C_{p\max}$ over the hemisphere follow the \cos^2 -law on the forward part of the hemisphere. Near the shoulder they agree with those calculated by assuming a Prandtl-Meyer expansion.
27. The wall temperature difference distributions exhibit a maximum at the 11-degree angular position. This maximum was found to disappear for the case of zero heat transfer.
28. The distributions of the local non-dimensional heat-transfer parameter show peak values for angular positions of about 11 degrees and 45 degrees, and high values at the shoulder.
29. The numerical results are about 20 percent larger than calculated for an isothermal body by the modified incompressible theory, given by Korobkin.

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APPENDIX A

Temperature interpolation
Sub/ (β_{12}/ν)

Temperature interpolation
NxD/ $(\beta D^2/\nu)$

x/D		p/p _{atm} = 0		T _w °K		ΔT°K		(T _e -T _w)/(T _e -T _w)		N _u _x		N _u _x / $\sqrt{Re_x}$	
0	1.000	393	28.9	1.012	0	.702**		0	1.000	402	35.7	1.014	0
.098		394	29.0	1.000	35.8	.736		.058	.463	35.0	1.006	45.7	.753
.164		390	26.7	1.000	55.1	.694		.161	398	33.0	1.000	71.2	.717
.257		386	24.8*	1.000	81.4	.724		.257	394	30.0*	1.003	72.5	.725
.302	.627			1.018	105	.775		.312	.626		1.022	140	.874
.393		381	20.0	1.018				.313	389	25.5			
.424	.390							.411					.730**
.518	.211	374	13.9*	1.067	100	.742		.518					
.547								.547	.211				
.658		368	8.6	1.169	83.1	.750		.658					
.720	.091	364	6.6	1.189	79.2/97.5	.911/1.12		.720	.6775				
.785								.785	.367	7.5	1.142	92.1/113	.842/1.040
.805	.0477	363	6.0	1.170	100	1.19		.805	.0467	366	6.5	1.164	110
.889	.0425							.889	.0415				
1.652	.0330	361*	3.6*	1.045	130	1.29		1.652	.0364				
1.848								1.848					
2.320	.0263	362	3.0	1.000	173	1.62		2.320	.0235	364	3.0	1.008	176
2.448								2.448	.0186				1.285
2.985	.0224							2.985					
x/D		p/p _{atm} = 0		T _w °K		ΔT°K		(T _e -T _w)/(T _e -T _w)		N _u _x		N _u _x / $\sqrt{Re_x}$	
0	1.000	394	34.5	1.014	0	.756**		0	1.000	408	39.7	1.014	0
.098		401	34.3	1.006	43.4*	.785		.098	.410	39.3	1.000	53.2	.817
.164		396	32.0	1.003	68.2	.740		.164	404	36.5	1.000	79.3	.748
.257		392	29.5*	1.003	100	.775		.257	401	33.7*	1.000	119	.796
.302	.627			1.024	132	.838		.362	.627		1.020	155	.842
.393		387	24.5	1.024				.393	394	27.6			
.424	.399							.424	.401				.782**
.518	.215	377	16.5*	1.060	121	.784		.518					
.547								.547	.212				
.658		369	9.7	1.172	94.2	.735		.658	.0759	374	11.3	1.018	114
.720	.0788							.720	.0785	370	7.5	1.266	94.2/117
.785		365	7.5	1.159	91.0/1.077			.785	.0460	370	6.9	1.179	120
.805	.0474							.805	.0460				1.04
.869		364	6.5	1.171	108	1.11		.869	.0410				
.879	.0422							.879	.0304				
1.652	.0365	361*	4.4*	1.053	159	1.35		1.652					
1.848								1.848					
2.320	.0228	363	3.2	1.004	186	1.49		2.320	.0235	365	2.7	1.000	159
2.448								2.448	.0186				1.08
2.985	.0186							2.985					

* Temperature interpolation
** $Nu_D / (\Delta D^2 / \rho_f)$

* Temperature interpolation
** $Nu_D / (\Delta D^2 / \rho_f)$

$T_{\infty} = 6.49$
 $T_0 = 600^{\circ}\text{K}$
 $Re_{\infty} = 1.51 \times 10^5$
 $P_0 = 7.38 \text{ ATM}$

$\frac{f(T_{\infty} - T_w)}{f_{D0}/U_{\infty}} = 1.25$
 $U_{\infty} = 1045 \text{ ft/sec}$

$\frac{f(T_{\infty} - T_w)}{f_{D0}/U_{\infty}} = .407$
 $Re_{\infty} = 2.37 \times 10^5$
 $P_0 = 11.57 \text{ ATM}$

x/D	$p/p_{\infty} = 0$	$T_w^{\circ}\text{K}$	$\Delta T^{\circ}\text{K}$	$(T_{\infty} - T_w) f_{eff - T_w}$	Nu_x	$Nu_x / \sqrt{Re_x}$
0	1.0000	369	54.4	1.009	0	.763**
.098	.370	53.5	1.000	.43.4	.774	.740**
.164	.364	49.5	1.004	.67.0	.735	.801
.257	.358	45.5*	1.004	.97.1	.737	.750
.302	.631	352	.39.4	1.039	131	.742
.393	.352	332	.23.0*	1.057	103	.634
.424	.395	323	.15.5	1.048	.619	.806
.518	.218	323	.15.5	.60.5	.643	.668
.547	.658	720	.0860	.786/.962	.726	.720
.720	.785	317	.12.3	1.059	.88.3/169	.785
.805	.0499	314	.8.8	1.068	.86.5	.865
.869	.0454	307	.2.1	.98.3	.86.5	.869
.898	.0322	307	.2.1	.98.3	.86.1	.853
1.652	.369*	307	.2.1	.1.013	.11.5	.652
1.848	.6261	307	.2.1	.98.3	.67.4	.848
2.320	.2.848	307	.2.1	.98.3	.41.2	.2.320
2.985	.0219	2.985	.2.985	.98.3	.2.985	.2.985

$T_{\infty} = 6.50$
 $T_0 = 600^{\circ}\text{K}$
 $Re_{\infty} = 1.51 \times 10^5$
 $P_0 = 5.52 \text{ ATM}$

$\frac{f(T_{\infty} - T_w)}{f_{D0}/U_{\infty}} = 1.25$
 $U_{\infty} = 1045 \text{ ft/sec}$

$\frac{f(T_{\infty} - T_w)}{f_{D0}/U_{\infty}} = .410$
 $U_{\infty} = 1045 \text{ ft/sec}$

x/D	$p/p_{\infty} = 0$	$T_w^{\circ}\text{K}$	$\Delta T^{\circ}\text{K}$	$(T_{\infty} - T_w) f_{eff - T_w}$	Nu_x	$Nu_x / \sqrt{Re_x}$
0	1.0000	374	57.5	1.009	0	.727**
.098	.376	53.9	1.000	.47.2	.737	.740**
.164	.370	50.0*	1.000	.74.6	.701	.801
.257	.364	50.0*	1.000	.10.8	.679	.750
.302	.635	357	.43.4	1.040	162	.742
.393	.357	335	.25.0*	1.048	111	.634
.424	.399	326	.17.6	1.049	103	.806
.518	.212	326	.17.6	.645	.645	.668
.547	.658	720	.0830	.752/.928	.726	.720
.720	.785	319	.13.4	1.060	.96.3/119	.785
.805	.0494	315	.9.8	1.069	.96.3	.865
.869	.0449	310*	.6.1*	1.012	.850	.850
.898	.0324	310*	.6.1*	1.012	.850	.850
1.652	.310*	310*	.6.1*	1.012	.850	.850
1.848	.6259	308	.2.4	.98.7	.79.0	.479
2.320	.2.848	308	.2.4	.98.7	.79.0	.525
2.985	.0217	2.985	.2.985	.98.7	.2.985	.2.985

$T_{\infty} = 6.48$
 $T_0 = 602^{\circ}\text{K}$
 $Re_{\infty} = 2.37 \times 10^5$
 $P_0 = 11.57 \text{ ATM}$

$\frac{f(T_{\infty} - T_w)}{f_{D0}/U_{\infty}} = 1.25$
 $U_{\infty} = 1042 \text{ ft/sec}$

$\frac{f(T_{\infty} - T_w)}{f_{D0}/U_{\infty}} = .405$
 $U_{\infty} = 1042 \text{ ft/sec}$

* Temperature interpolation
** $Nu_x / (f_{D0}^2 / \rho)$ interpolation

* Temperature interpolation
** $Nu_x / (f_{D0}^2 / \rho)$ interpolation

$\frac{f_{D}(T_e - T_w)}{f_{D}(T_e - T_w)} = 1.268$							
$\frac{f_{D}(T_e - T_w)}{f_{D}(T_e - T_w)} = 1.265$							
$\frac{f_{D}(T_e - T_w)}{f_{D}(T_e - T_w)} = 1.265$ <th data-kind="ghost"></th>							
$\frac{f_{D}(T_e - T_w)}{f_{D}(T_e - T_w)} = 1.265$ <th data-kind="ghost"></th>							
x/D	p/p _{atm} = 0	ΔT_w /K	T_w /K	$f_{D}(T_e - T_w) \cdot f_{eff} - f_{D}$	Nu _x	$Nu_x / \sqrt{Re_x}$	
0	1.000	36.9	72.5	1.010	0	54.8**	
.098	37.0	76.8	1.007	49.7	.724		
.164	35.8	80.5	.997	88.3	.776		
.257	34.6	76.0*	1.000	126	.760		
.302	.627	33.0	65.9	1.017	170	.824	
.393	.396	30.0	48.5*	1.019	163	.755	
.424	.518	.211	32.0	1.034	140	.730	
.658	.720	.0777	31.6	1.034	153/188	.992/1.22	
.785	.805	.0450	31.9	1.049	212	1.41	
.869	.898	.0405	1.848	1.017	249	1.52	
1.552	1.820	.0318	247*	19.5*	1.017	2.99	
2.320	2.848	.0238	237	6.8	1.000	1.63	
2.985	.0198	.0198					

$\frac{f_{D}(T_e - T_w)}{f_{D}(T_e - T_w)} = 1.265$							
$\frac{f_{D}(T_e - T_w)}{f_{D}(T_e - T_w)} = 1.25$							
$\frac{f_{D}(T_e - T_w)}{f_{D}(T_e - T_w)} = 1.25$							
$\frac{f_{D}(T_e - T_w)}{f_{D}(T_e - T_w)} = 1.25$							
x/D	p/p _{atm} = 0	ΔT_w /K	T_w /K	ΔT_w /K	$f_{D}(T_e - T_w) \cdot f_{eff} - f_{D}$	Nu _x	$Nu_x / \sqrt{Re_x}$
0	1.000	38.7	82.4	1.011	0	712**	
.098	38.9	87.9	1.004	59.3	.785		
.164	37.2	89.6	1.003	66.1	.777		
.257	35.8	74.1*	1.000	143	.786		
.302	.627	34.1	72.8	1.017	263	.837	
.393	.396	30.2	53.3*	1.019	183	.736	
.424	.518	.212	36.5	1.035	163	.773	
.658	.720	.0749	34.5	1.038	184/226	1.084/1.337	
.785	.805	.0456	26.9	1.054	256	1.550	
.869	.898	.0405	1.848	1.017	278	1.139	
1.552	1.820	.0314	247*	18.0*	1.000	197	
2.320	2.848	.0243	241	8.1			
2.985	.0201	.0201					

* Temperature Interpolation
** $Nu_D / (\beta D^2 / \nu)$ Interpolation

* Temperature Interpolation
** $Nu_D / (\beta D^2 / \nu)$ Interpolation

$\frac{(\Gamma_e - \Gamma_w)}{\Gamma_e} = 0.362$							$\frac{(\Gamma_e - \Gamma_w)}{\Gamma_e} = 0.360$							$\frac{(\Gamma_e - \Gamma_w)}{\Gamma_e} = 0.345$								
$\frac{M_{\infty}}{A_D} = 8.01$				$\frac{M_{\infty}}{A_D} = 8.00$			$\frac{M_{\infty}}{A_D} = 8.00$				$\frac{M_{\infty}}{A_D} = 8.00$			$\frac{M_{\infty}}{A_D} = 7.99$				$\frac{M_{\infty}}{A_D} = 7.99$				
$T_0 = 6450K$				$T_0 = 6450K$			$T_0 = 6450K$				$T_0 = 6410K$			$T_0 = 6410K$				$T_0 = 6410K$				
$Re_{\infty} = 1.58 \times 10^5$				$Re_{\infty} = 1.59 \times 10^5$			$Re_{\infty} = 1.60 \times 10^5$				$Re_{\infty} = 1.61 \times 10^5$			$Re_{\infty} = 1.62 \times 10^5$				$Re_{\infty} = 1.63 \times 10^5$				
$P_0 = 15.00 \text{ ATM}$				$P_0 = 15.00 \text{ ATM}$			$P_0 = 15.00 \text{ ATM}$				$P_0 = 15.00 \text{ ATM}$			$P_0 = 15.00 \text{ ATM}$				$P_0 = 15.00 \text{ ATM}$				
x/D	p/p _∞ = 0	$\tau_w \frac{\partial}{\partial x}$	$\Delta \tau^K$	$(\Gamma_e - \Gamma_w) \frac{\partial \Gamma_{eff} - \Gamma_w}{\partial x}$				ν_{∞}	$\nu_{\infty} / \sqrt{\Gamma_{\infty}}$				ν_{∞}	$(\Gamma_e - \Gamma_w) \frac{\partial \Gamma_{eff} - \Gamma_w}{\partial x}$				ν_{∞}	$\nu_{\infty} / \sqrt{\Gamma_{\infty}}$			
0	1.000	.420	.46.5	1.018	0	.731**	0	1.000	1.000	.426	.48.5	1.019	0	.713**	0	.426	.46.7	1.013	.424	.55.0	.867	.780
.098	.425	.51.3	1.011	.790	.096	.16.4	.096	.425	.424	.424	.48.5	.005	.425	.005	.424	.424	.48.5	.005	.424	.55.0	.69.6	.757
.164	.414	.41.8	1.002	.57.4	.716	.257	.716	.416	.43.5*	.416	.43.5*	.97.7	.97.7	.97.7	.97.7	.97.7	.97.7	.97.7	.97.7	.97.7	.97.7	.97.7
.257	.408	.37.5*	1.003	.81.2	.716	.362	.647	.408	.38.2	.408	.38.2	.135	.135	.135	.135	.135	.135	.135	.135	.135	.135	.135
.302	.639	.402	.34.5	1.028	.117	.393	.424	.424	.424	.424	.424	.105	.105	.105	.105	.105	.105	.105	.105	.105	.105	.105
.393	.409	.387	.22.2*	1.035	.100	.740	.518	.518	.518	.518	.518	.293	.293	.293	.293	.293	.293	.293	.293	.293	.293	.293
.424	.518	.208	.22.2*	1.035	.100	.740	.547	.547	.547	.547	.547	.381	.381	.381	.381	.381	.381	.381	.381	.381	.381	.381
.547	.658	.376	.13.3	1.030	.79.0	.681	.658	.658	.658	.658	.658	.375	.375	.375	.375	.375	.375	.375	.375	.375	.375	.375
.720	.702	.0826	.372	9.8	1.004	.72.0/88.6	.739/.969	.785	.785	.785	.785	.993	.993	.993	.993	.993	.993	.993	.993	.993	.993	.993
.785	.805	.0566	.369	7.3	1.006	.73.6	.785	.785	.785	.785	.785	.373	.373	.373	.373	.373	.373	.373	.373	.373	.373	.373
.869	.898	.0480	.0346	.367*	5.5*	1.000	119	1.006	1.006	1.006	1.006	.0421	.0421	.0421	.0421	.0421	.0421	.0421	.0421	.0421	.0421	.0421
1.652	1.848	.0371	.363	1.8	1.021	58.9	.385	.385	.385	.385	.385	.0278	.0278	.0278	.0278	.0278	.0278	.0278	.0278	.0278	.0278	.0278
2.320	2.848	.0371	.363	1.8	1.021	58.9	.385	.385	.385	.385	.385	.0249	.0249	.0249	.0249	.0249	.0249	.0249	.0249	.0249	.0249	.0249
2.985	.0371											.0307	.0307	.0307	.0307	.0307	.0307	.0307	.0307	.0307	.0307	.0307
x/D	p/p _∞ = 0	$\tau_w \frac{\partial}{\partial x}$	$\Delta \tau^K$	$(\Gamma_e - \Gamma_w) \frac{\partial \Gamma_{eff} - \Gamma_w}{\partial x}$				ν_{∞}	$\nu_{\infty} / \sqrt{\Gamma_{\infty}}$				ν_{∞}	$(\Gamma_e - \Gamma_w) \frac{\partial \Gamma_{eff} - \Gamma_w}{\partial x}$				ν_{∞}	$\nu_{\infty} / \sqrt{\Gamma_{\infty}}$			
0	1.000	.424	.49.4	1.018	0	.809**	0	1.000	1.000	.431	.51.9	1.014	0	.716**	0	.431	.51.9	1.010	.436	.58.0	.848	.767
.098	.429	.52.9	1.014	.46.4	.63.4	.786	.658	.658	.658	.658	.658	.429	.429	.429	.429	.429	.429	.429	.429	.429	.429	.429
.164	.418	.45.4	1.004	.63.1	.88.0	.758	.758	.758	.758	.758	.758	.422	.422	.422	.422	.422	.422	.422	.422	.422	.422	.422
.257	.411	.40.5*	1.006	.40.5*	.88.0	.88.0	.88.0	.88.0	.88.0	.88.0	.88.0	.392	.634	.392	.634	.392	.634	.392	.634	.392	.634	.392
.302	.637	.404	.36.2	1.031	.124	.871	.871	.871	.871	.871	.871	.413	.42.3	.413	.42.3	.413	.42.3	.413	.42.3	.413	.42.3	.413
.393	.414	.389	.24.0*	1.033	.110	.774	.518	.518	.518	.518	.518	.397	.23.5*	.397	.23.5*	.397	.23.5*	.397	.23.5*	.397	.23.5*	.397
.547	.215	.378	14.8	1.023	89.0	.735	.547	.547	.547	.547	.547	.333	.18.3	.333	.18.3	.333	.18.3	.333	.18.3	.333	.18.3	.333
.658	.720	.0810	.372	10.2	.599	75.3/92.9	.762/.939	.785	.785	.785	.785	.376	.12.0	.376	.12.0	.376	.12.0	.376	.12.0	.376	.12.0	.376
.785	.805	.0533	.372	9.7	.595	99.0	1.05	.595	.595	.595	.595	.0506	.0506	.0506	.0506	.0506	.0506	.0506	.0506	.0506	.0506	.0506
.869	.903	.0435	.372	9.7	.595	99.0	1.05	.595	.595	.595	.595	.0421	.0421	.0421	.0421	.0421	.0421	.0421	.0421	.0421	.0421	.0421
1.652	1.848	.0298	.365*	3.1*	.591	69.0	.610	.591	.591	.591	.591	.0275	.0275	.0275	.0275	.0275	.0275	.0275	.0275	.0275	.0275	.0275
2.320	2.848	.0302	.362	0.5	1.009	16.5	.105	.105	.105	.105	.105	.0212	.0212	.0212	.0212	.0212	.0212	.0212	.0212	.0212	.0212	.0212
2.985	.0340											.365	.1.8	.365	.1.8	.365	.1.8	.365	.1.8	.365	.1.8	.365

$\Delta D = 8.00$
 $T_0 = 664.50K$
 $Re_D = 9.13 \times 10^5$
 $P_0 = 9.15 \text{ ATM}$

$\left[\frac{(T_e - T_w)}{T_0} \right]_m = .462$
 $\frac{\Delta D}{U_0 D} = 1.13$
 $U_0 D = 1113 \text{ in/sec}$

x/D	p/p _{atm} = 0	T _w °K	ΔT°K	(T _e -T _w)Δe _{eff} (T _w)	Nu _x	Nu _x / $\sqrt{Re_x}$
0	1.000	376.5	57.5	1.007	0	.97***
.098	377	56.7	1.010	36.7	.998	.981
.164	365	51.5	1.000	.777	.164	.780
.257	357.5	41.0*	1.003	.765	.257	.741
.302	646	38.0	1.011	.910	.302	.880
.393	351	38.0	1.024	.89.7	.393	.763
.421	334	24.5*	1.024	.755	.421	.726
.518	321.5	14.5	1.013	.71.2	.518	.843/1.045
.517	321.5	14.5	1.013	.71.2	.547	
.638	321.5	12.0	1.018	69.5/65.4	.638	
.720	.0955	316.5	12.0	7.39/4.570	.785	
.785	.0648	321	16.8	1.013	.865	
.869	.0568	313.5*	10.0*	1.013	.866	
.898	.0486	313.5*	10.0*	1.013	.898	
1.652	.0320	308	4.0	1.022	.652	
1.848	.0511	308	4.0	1.03	.848	
2.320	.0443	2.985	.0477	.650	.2320	
2.985					.2.985	

$\Delta D = 8.05$
 $T_0 = 660.40K$
 $Re_D = 1.55 \times 10^5$
 $P_0 = 15.95 \text{ ATM}$

$\left[\frac{(T_e - T_w)}{T_0} \right]_m = .453$
 $\frac{\Delta D}{U_0 D} = 1.185$
 $U_0 D = 1122 \text{ in/sec}$

x/D	p/p _{atm} = 0	T _w °K	ΔT°K	(T _e -T _w)Δe _{eff} (T _w)	Nu _x	Nu _x / $\sqrt{Re_x}$
0	1.000	387	63.5	1.007	0	.869***
.098	390	63.7	1.010	13.5	.354	.0
.164	365	56.5	1.003	12.2	.725	.52.4
.257	371	50.3*	1.012	87.2	.257	.856
.302	.540	46.5	1.010	126	.362	.761
.393	.400	34.1	1.024	113	.393	.759
.424	.518	31.5*	1.024	113	.424	
.518	.547	21.2	1.012	87.1	.518	.838
.658	.328	19.0	1.012	.720	.547	.764
.720	.6729	221	15.0	1.005	.720	.727
.785	.0518	323	17.7	1.004	.785	.818/1.009
.805	.0518	323	17.7	1.004	.805	.0
.869	.0435	318	13.0*	.997	.869	.0
.898	.0318	318	13.0*	.997	.898	
1.652	.0318	311	5.9	.950	.1.652	
2.320	.0303	311	5.9	.950	.2.320	
2.848	.0284				.2.848	
2.985					.2.985	

$\Delta D = 8.04$
 $T_0 = 669.0K$
 $Re_D = 2.01 \times 10^5$
 $P_0 = 20.35 \text{ ATM}$

$\left[\frac{(T_e - T_w)}{T_0} \right]_m = .446$
 $\frac{\Delta D}{U_0 D} = 1.19$
 $U_0 D = 1116 \text{ in/sec}$

x/D	p/p _{atm} = 0	T _w °K	ΔT°K	(T _e -T _w)Δe _{eff} (T _w)	Nu _x	Nu _x / $\sqrt{Re_x}$
0	1.000	393	67.2	1.000	0	.793**
.098	395	70.5	1.010	389	.098	.781
.164	364	51.5	1.000	.777	.164	.780
.257	357.5	41.0*	1.003	.765	.257	.741
.302	646	38.0	1.011	.910	.302	.880
.393	351	38.0	1.024	.89.7	.393	.763
.421	334	24.5*	1.024	.755	.421	.726
.518	321.5	14.5	1.013	.71.2	.518	.843/1.045
.517	321.5	14.5	1.013	.71.2	.547	
.638	321.5	12.0	1.018	69.5/65.4	.638	
.720	.0955	316.5	12.0	7.39/4.570	.785	
.785	.0648	321	16.8	1.013	.865	
.869	.0568	313.5*	10.0*	1.013	.866	
.898	.0486	313.5*	10.0*	1.013	.898	
1.652	.0320	308	4.0	1.022	.652	
1.848	.0511	308	4.0	1.03	.848	
2.320	.0443	2.985	.0477	.650	.2320	
2.985					.2.985	

$\Delta D = 8.04$
 $T_0 = 670K$
 $Re_D = 2.39 \times 10^5$
 $P_0 = 23.6 \text{ ATM}$

$\left[\frac{(T_e - T_w)}{T_0} \right]_m = .440$
 $\frac{\Delta D}{U_0 D} = 1.19$
 $U_0 D = 1117 \text{ in/sec}$

* Temperature Interpolation
** Nu_x/($\Delta D/U_0 D$)^{1/2}

$\frac{N_{D0}}{N_{D0}} = 7.93$							$\frac{[(T_e - T_w)/T_w]}{A/D} = 0.429$						
$T_0 = 6720K$							$\frac{[(T_e - T_w)/T_w]}{A/D} = 0.429$						
$Re_{D0} = 2.86 \times 10^5$							$\frac{[(T_e - T_w)/T_w]}{A/D} = 0.429$						
$P_0 = 28.0 \text{ ATM}$							$\frac{[(T_e - T_w)/T_w]}{A/D} = 0.429$						
x/D	p/p _{atm} = 0	T _w °K	Δ T°K	T _w °K	(T _e - T _w)Δ T°K	Nu _x	x/D	p/p _{atm} = 0	T _w °K	Δ T°K	(T _e - T _w)Δ T°K	Nu _x	Nu _x / $\sqrt{Re_x}$
0	1.000	413	78.0	1.008	0	.824**	0	1.000	370	72.0	1.011	0	.758**
.098		419	78.9	1.012	.563	.990	.098		375	71.8	1.008	42.7	.798
.164		403	70.5	1.004	.817	.767	.164		358	64.6	1.016	64.1	.710
.257		393	65.5*	1.004	119	.770	.257		344	56.5*	1.032	85.5	.640
.302		640					.302		636				
.393		382	59.6	1.010	168	.873	.393		325	48.1	1.037	107	.641
.424		.413					.424		.406				
.518		358	40.4*	1.018	150	.761	.518		300	34.4*	1.045	107	.607
.547		.224					.547		.216				
.658		339	26.2	1.021	126	.738	.658		277	22.9	1.060	92.2	.566
.720		.0782					.720		.0841				
.785		329	20.4	1.007	118/146	.834/1.032	.785		.259	24.1	1.068	105/143	.798/.991
.805		.0462					.805		.0668				
.869		327	18.3	1.003	135	1.018	.869		.254	19.2	1.060	127	.904
.898		.0424					.898		.0506				
1.652		.0291					1.652		.0552				
1.848		319*	12.0*				1.290		1.848				
2.320		.0224					205		1.290				
2.888		316	8.9				1.983		1.368				
2.985		.0195					2.985		2.320				
									.0326				
									.991				
									2.6				
										.991			
										57.6			
											.270		

$\frac{N_{D0}}{N_{D0}} = 7.90$							$\frac{[(T_e - T_w)/T_w]}{A/D} = 0.566$						
$T_0 = 6660K$							$\frac{[(T_e - T_w)/T_w]}{A/D} = 0.566$						
$Re_{D0} = 9.92 \times 10^5$							$\frac{[(T_e - T_w)/T_w]}{A/D} = 0.566$						
$P_0 = 9.23 \text{ ATM}$							$\frac{[(T_e - T_w)/T_w]}{A/D} = 0.566$						
x/D	p/p _{atm} = 0	T _w °K	Δ T°K	T _w °K	(T _e - T _w)Δ T°K	Nu _x	x/D	p/p _{atm} = 0	T _w °K	Δ T°K	(T _e - T _w)Δ T°K	Nu _x	Nu _x / $\sqrt{Re_x}$
0	1.000	340	57.9	1.008	0	.652**	0	1.000	373	70.1	1.010	0	.683**
.098		344	58.3	1.006	.34.9	.785	.098		381	73.0	1.006	42.4	.748
.164		331	53.2	1.017	.656	.656	.164		363	62.6	1.017	59.9	.631
.257		318	46.3*	1.034	66.7	.606	.257		350	50.8*	1.036	91.1	.660
.302		638					.302		.638				
.393		302	38.1	1.039	85.2	.608	.393		329	50.5	1.041	115	.634
.424		.516					.424		.401				
.518		281	27.3*	1.052	81.9	.563	.518		305	38.3*	1.051	117	.618
.547		.227					.547		.217				
.658		263	17.4	1.069	65.3	.471	.658		282	26.6	1.058	106	.640
.720		.120					.720		.0802				
.785		249	18.4	1.079	86.0/105	.714/.876	.785		.261	23.8	1.062	113/139	.729/.896
.805		.0970					.805		.0502				
.869		244	14.1	1.070	90.6	.721	.869		.256	20.7	1.055	134	.869
.898		.0652					.898		.0430				
1.652							1.652		.0393				
1.848							1.848						
2.320		.0458					2.320		.0267				
2.848							2.848		.234				
2.985		.0403					2.985		.0286				

* Temperature Interpolation
** $Nu_D / (\beta D^2 / \rho^2)$ Interpolation

* Temperature Interpolation
** $Nu_D / (\beta D^2 / \rho^2)$ Interpolation

$\lambda_{\text{Op}} = 7.95$
 $T_{\text{O}} = 6840\text{K}$
 $\text{Re}_{\text{Op}} = 2.15 \times 10^5$
 $P_{\text{O}} = 23.3 \text{ ATM}$

$\frac{[(T_{\text{O}} - T_{\text{W}})/T_{\text{O}}]_{\text{Op}}}{\beta_{\text{Op}}/U_{\text{Op}} = 1.15}$
 $U_{\text{Op}} = 11.10 \text{ ft/sec}$

x/D	$p/p_{\infty} - 0$	T_w°	$\Delta T^{\circ}\text{K}$	$(T_{\text{O}} - T_w)\beta_{\text{eff}}(T_{\text{O}} - T_w)$	Nu_x	$Nu_x/\sqrt{Re_x}$
0	1.000	388	78.2	1.011	0	.727**
.098		395	85.5	1.008	.838	
.164		377	72.9	1.017	.667	
.257	.647	362	66.7*	1.038	1.03	.683
.302		339	56.0	1.042	1.25	
.393		313	41.3*	1.053	1.28	.669
.424		286	27.3	1.059	1.06	.634
.518		266	27.3	1.053	1.27/157	.569
.547		266	27.3	1.053	1.27/157	.569
.658		266	27.3	1.053	1.27/157	.569
.720		266	27.3	1.053	1.27/157	.569
.785		266	27.3	1.053	1.27/157	.569
.869		260	23.2	1.035	1.50	1.017
.898		260	23.2	1.035	1.50	1.017
.915		243*	9.5*	1.018	1.32	.685
1.652	.0328	243*	9.5*	1.018	1.32	.685
1.848	.0271	236	5.3	.957	1.13	.662
2.320		236	5.3	.957	1.13	.662
2.848		236	5.3	.957	1.13	.662
2.985		236	5.3	.957	1.13	.662

$\lambda_{\text{Op}} = 7.90$
 $T_{\text{O}} = 6840\text{K}$
 $\text{Re}_{\text{Op}} = 2.74 \times 10^5$
 $P_{\text{O}} = 27.5 \text{ ATM}$

$\frac{[(T_{\text{O}} - T_w)/T_{\text{O}}]_{\text{Op}}}{\beta_{\text{Op}}/U_{\text{Op}} = 1.15}$
 $U_{\text{Op}} = 11.28 \text{ ft/sec}$

x/D	$p/p_{\infty} - 0$	T_w°	$\Delta T^{\circ}\text{K}$	$(T_{\text{O}} - T_w)\beta_{\text{eff}}(T_{\text{O}} - T_w)$	Nu_x	$Nu_x/\sqrt{Re_x}$
0	1.000	395	81.5	1.012	0	.751**
.098		405	86.5	1.004	.56.4	.667
.164		387	76.0	1.021	.80.1	.723
.257	.638	370	69.1*	1.026	1.14	.698
.302		347	56.5	1.039	1.45	.690
.393		318	42.8*	1.053	1.40	.677
.424		318	42.8*	1.053	1.40	.677
.518		221	29.8	1.063	1.24	.625
.658		291	29.8	1.063	1.24	.625
.720	.0766	269	28.6	1.056	1.42/175	.878/1.081
.785	.0467	263	24.0	1.050	1.63	1.025
.869	.0407	263	24.0	1.050	1.63	1.025
.898		245*	10.0*	1.008	1.45	.732
1.652	.0302	245*	10.0*	1.008	1.45	.732
1.848	.0214	239	5.5	.990	1.24	.523
2.320		239	5.5	.990	1.24	.523
2.848		239	5.5	.990	1.24	.523
2.985		239	5.5	.990	1.24	.523

* Temperature Interpolation
** $Nu_x/\sqrt{Re_x}$

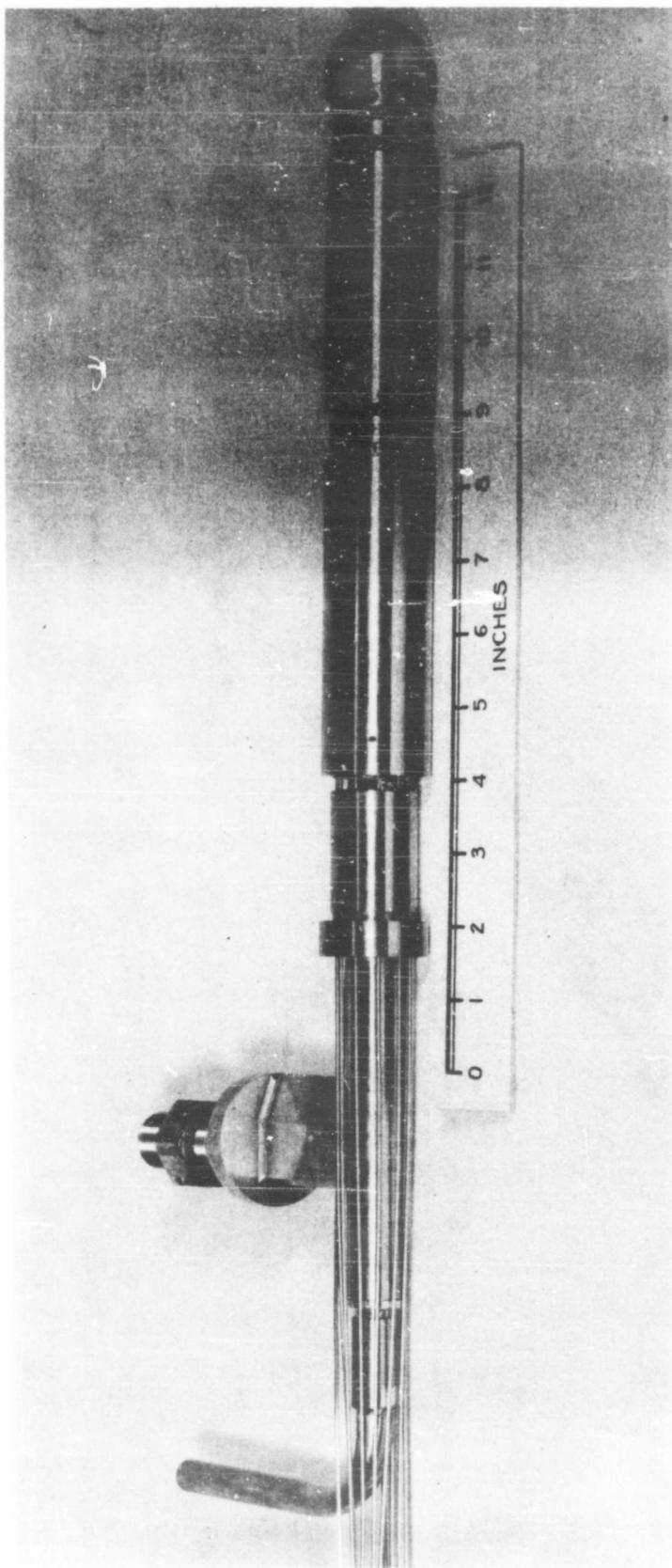


FIG. I HEMISPHERE - CYLINDER PRESSURE MODEL

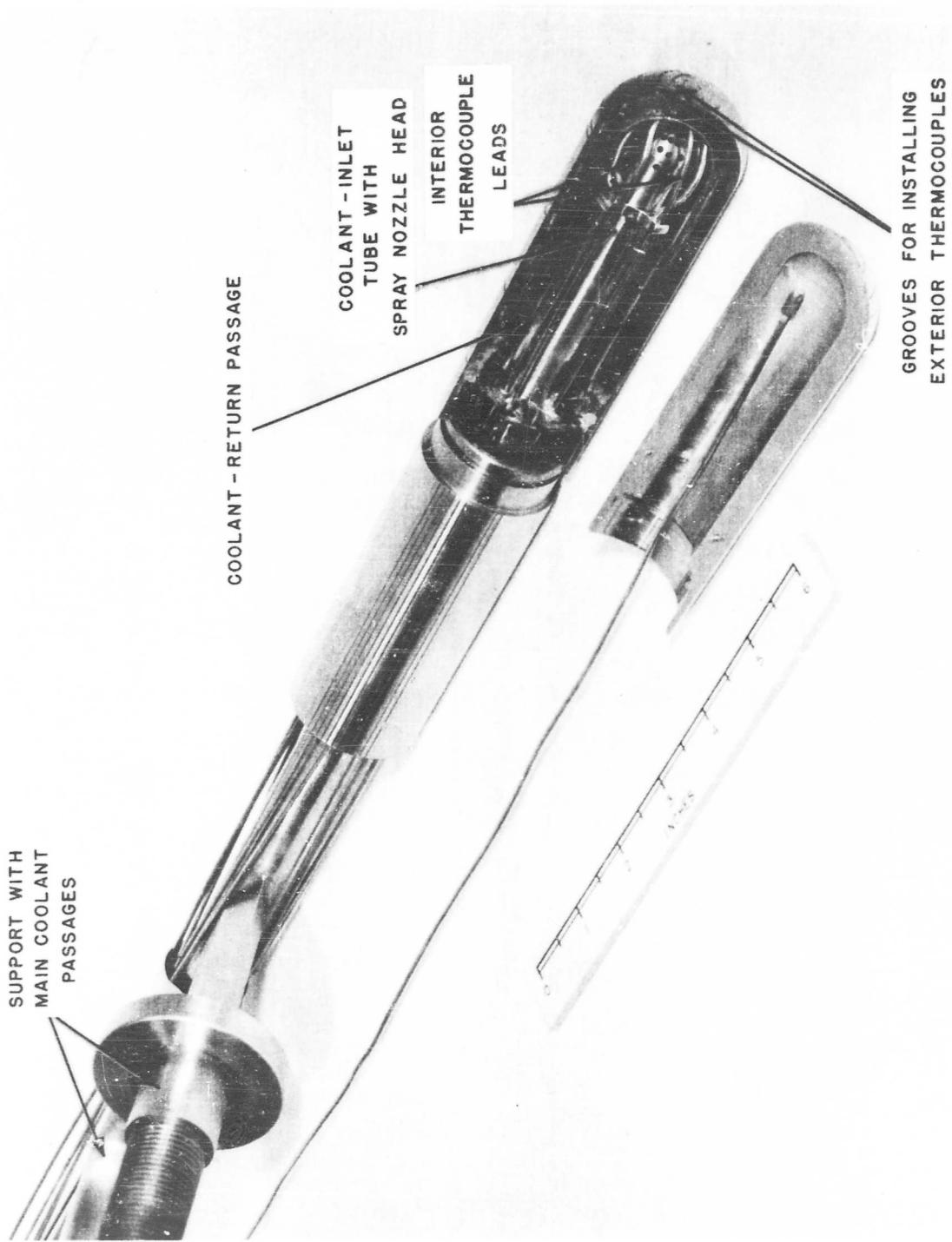


FIG. 2 CUT - AWAY VIEW OF HEMISPHERE - CYLINDER HEAT TRANSFER MODEL

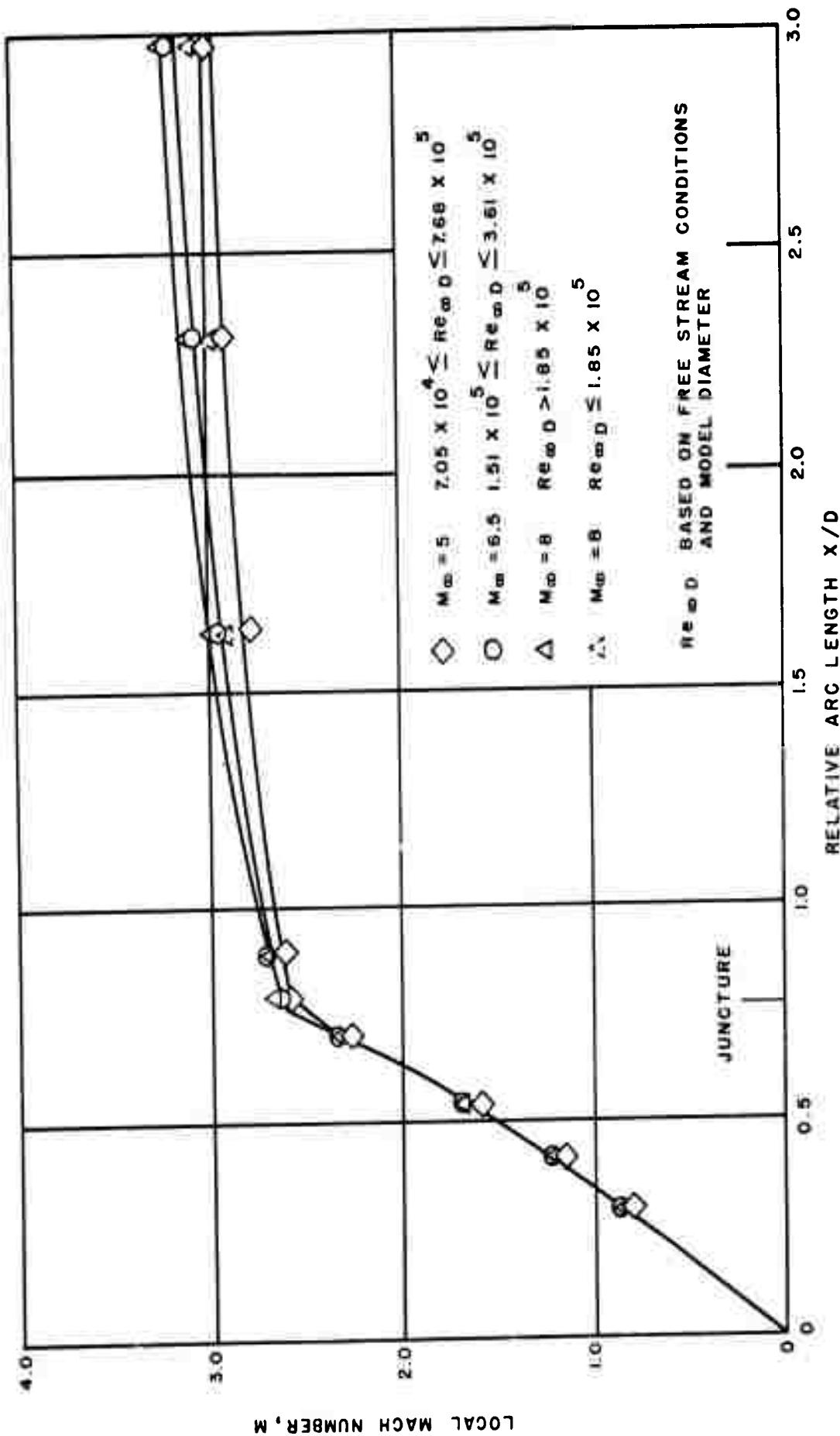


FIG. 3 MACH NUMBER DISTRIBUTION OVER HEMISPHERE CYLINDER

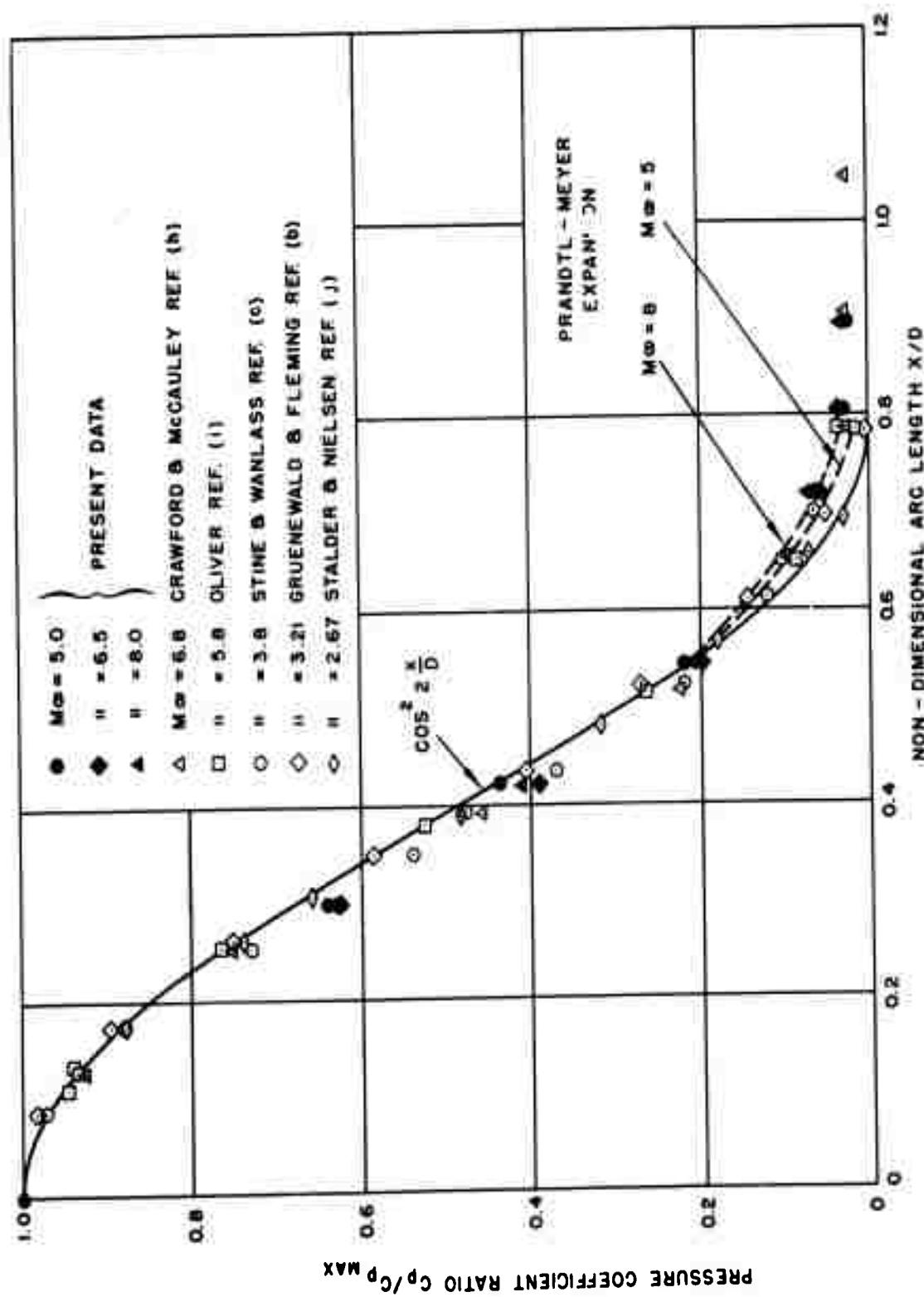


FIG. 4 PRESSURE COEFFICIENT DISTRIBUTION OVER HEMISPHERE-CYLINDER
AT VARIOUS MACH NUMBERS

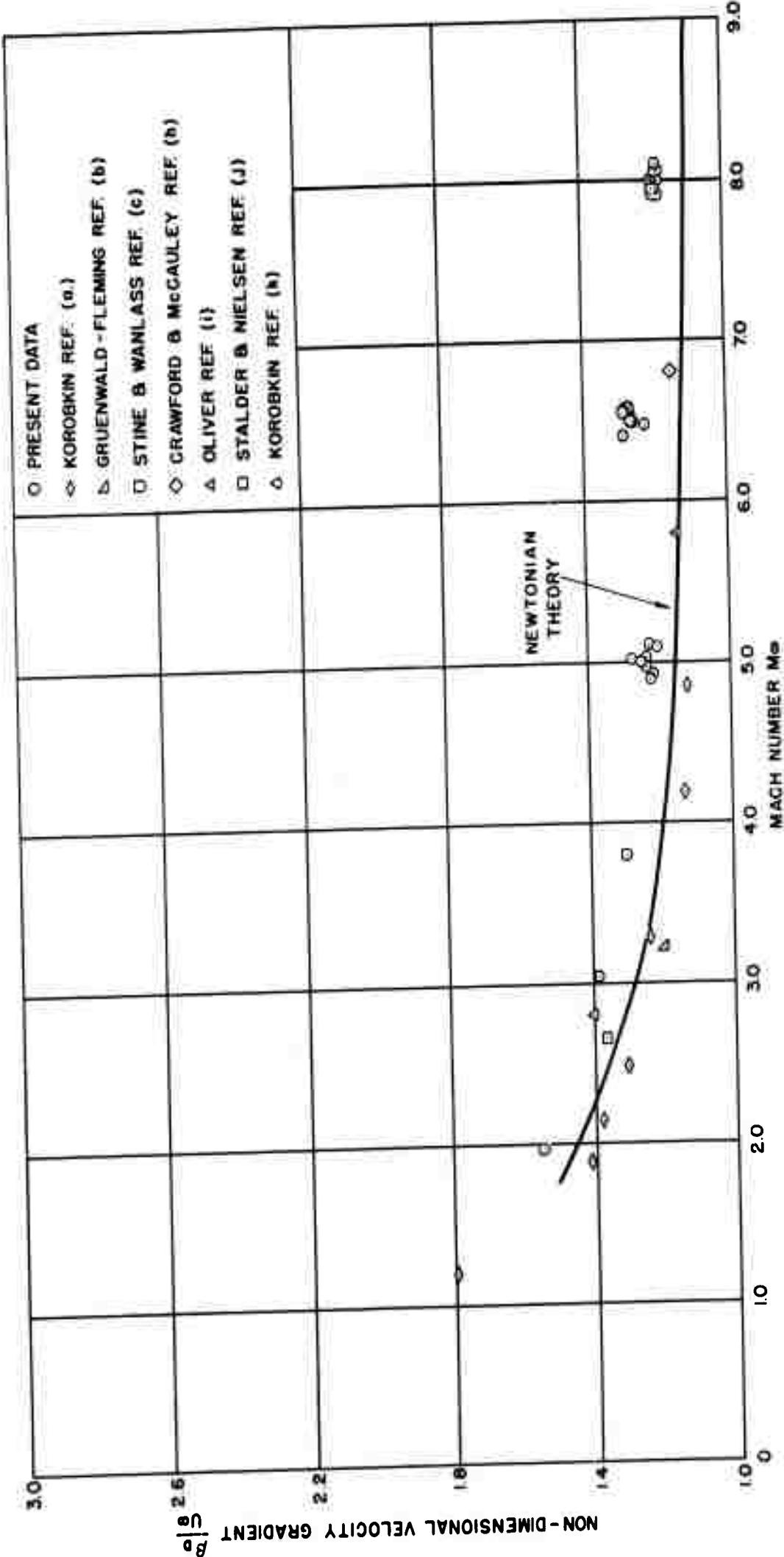


FIG. 5 NON-DIMENSIONAL VELOCITY GRADIENT AT MODEL STAGNATION POINT
VS FREE-STREAM MACH NUMBER

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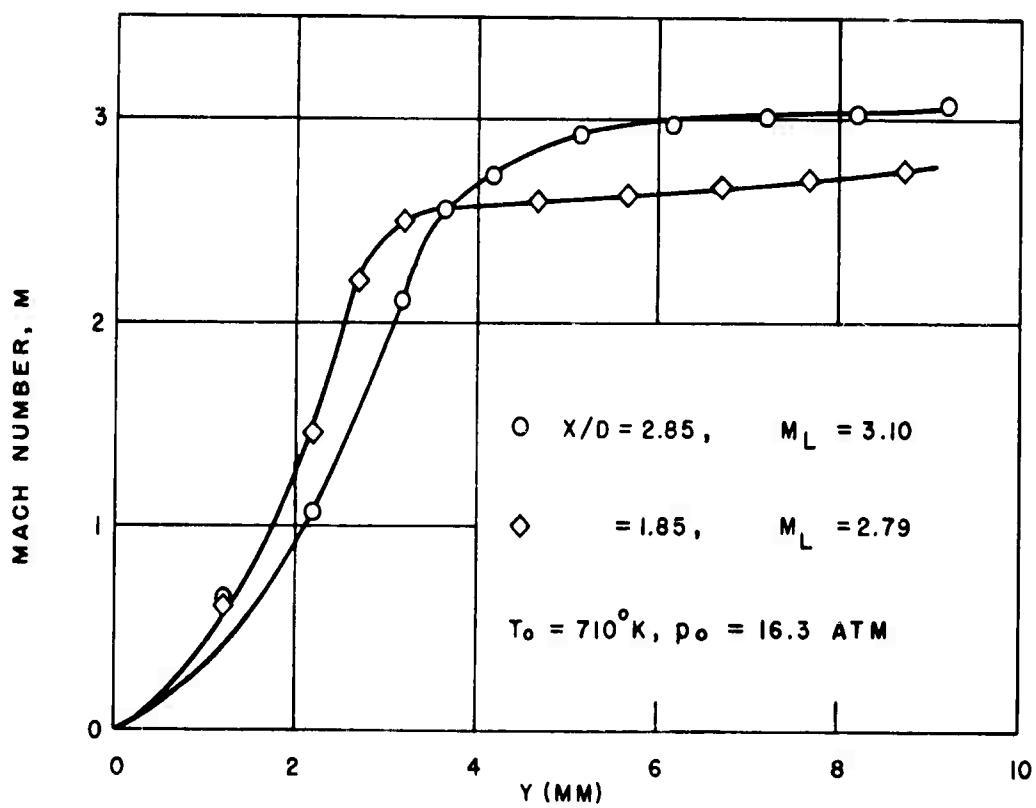
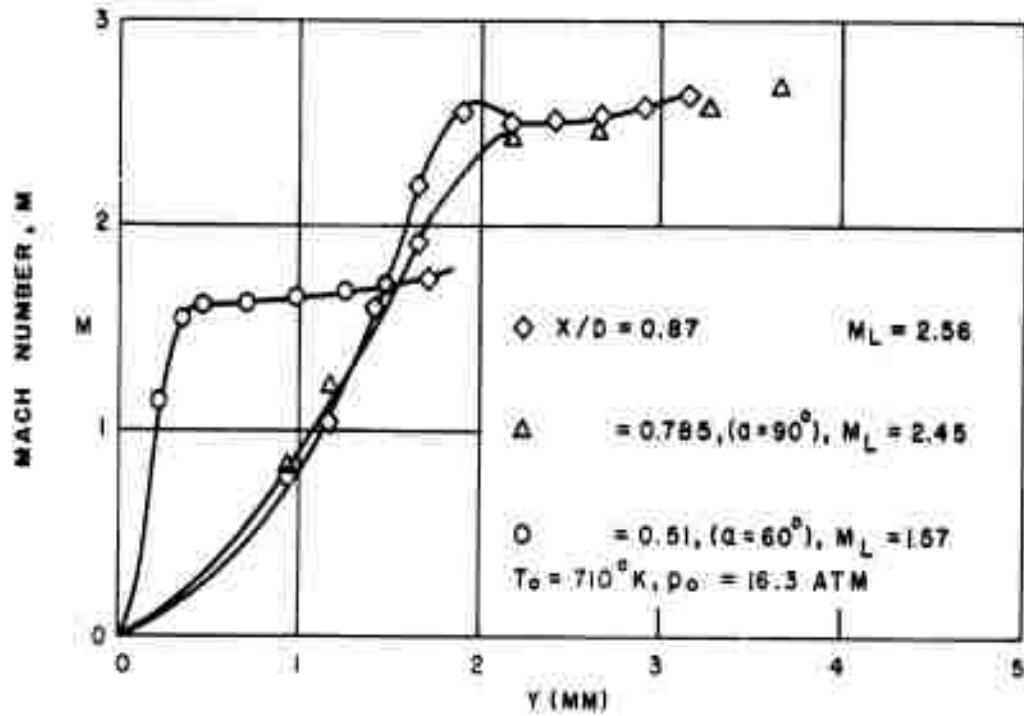


FIG. 6 MACH NUMBER DISTRIBUTION ACROSS
BOUNDARY LAYER AT VARIOUS STATIONS
ON THE HEMISPHERE-CYLINDER FOR A
FREE-STREAM MACH NUMBER OF 8

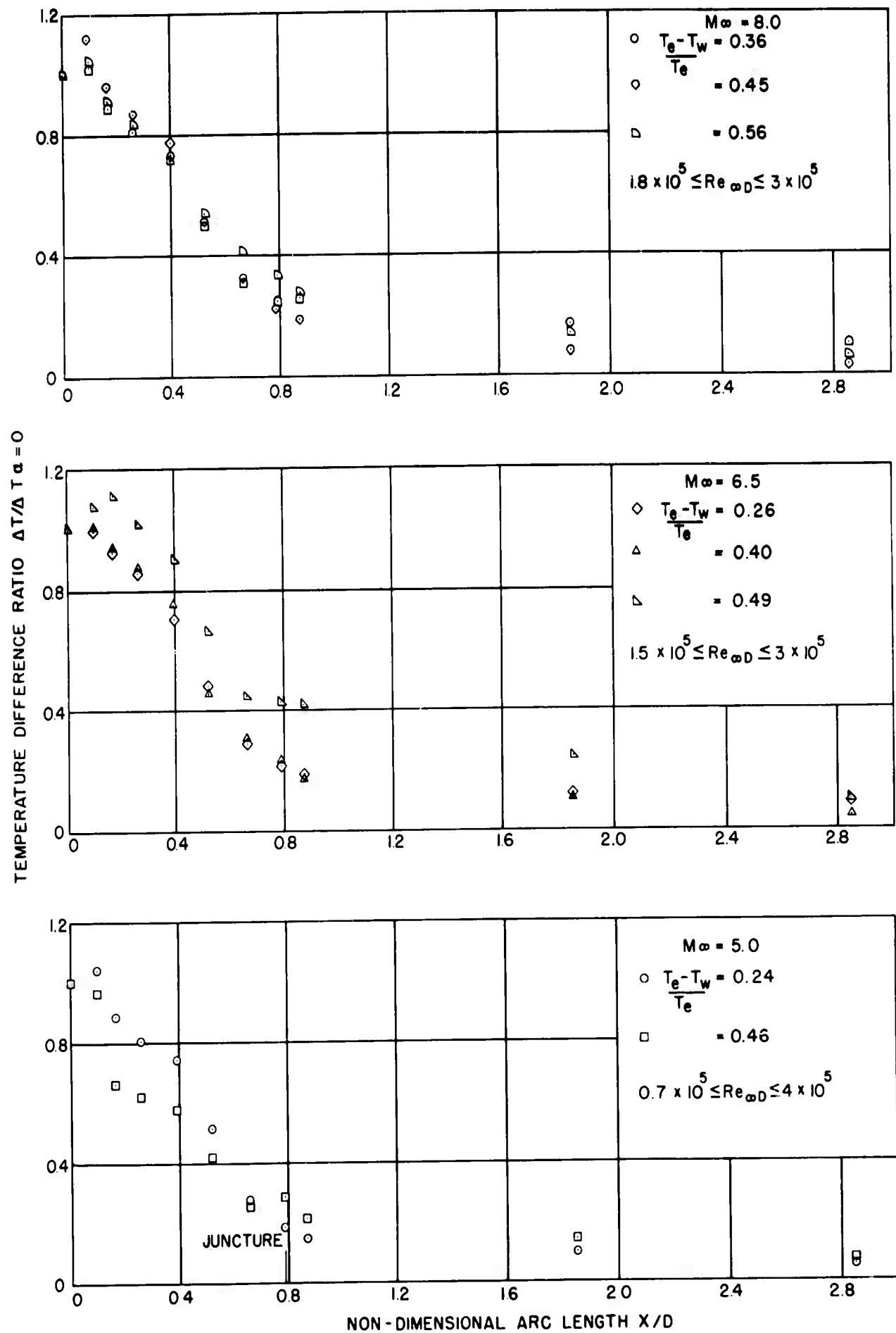


FIG. 7 VARIATION OF TEMPERATURE DIFFERENCE RATIO OVER HEMISPHERE-CYLINDER

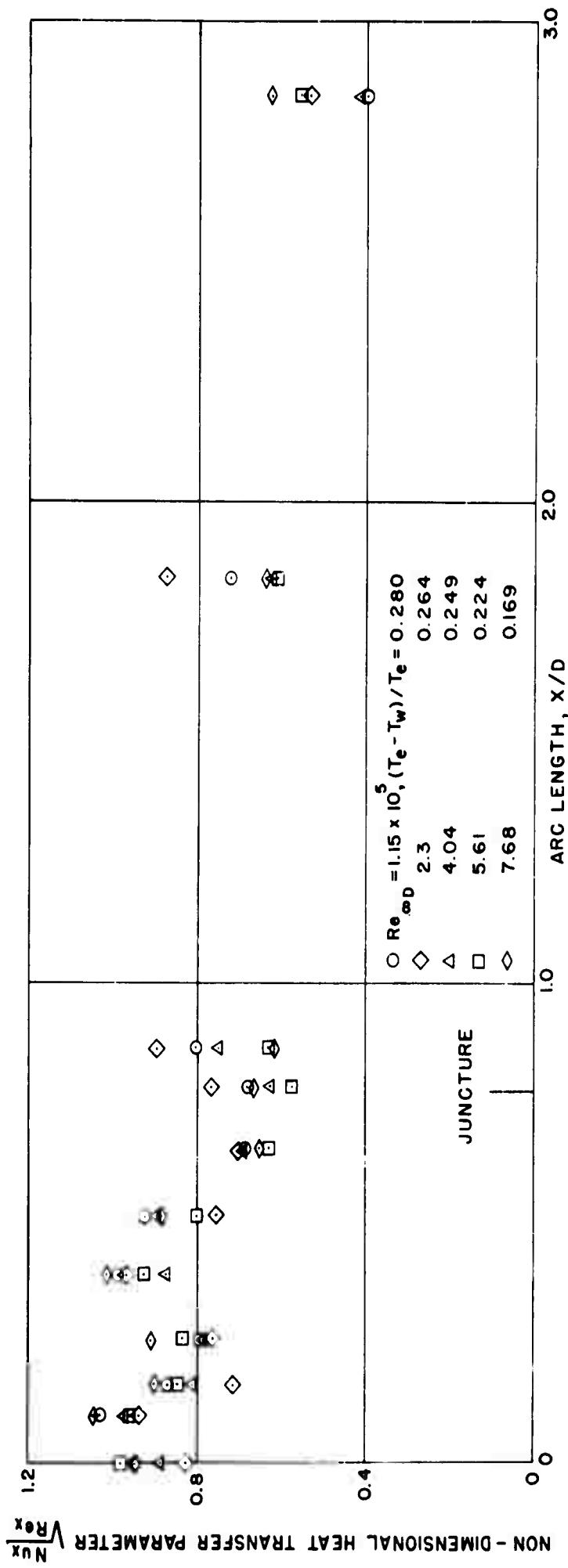


FIG. 8a NON-DIMENSIONAL HEAT-TRANSFER-PARAMETER DISTRIBUTION OVER
HEMISPHERE - CYLINDER

MACH NUMBER AND $T_w / T_e \sim 0.725$

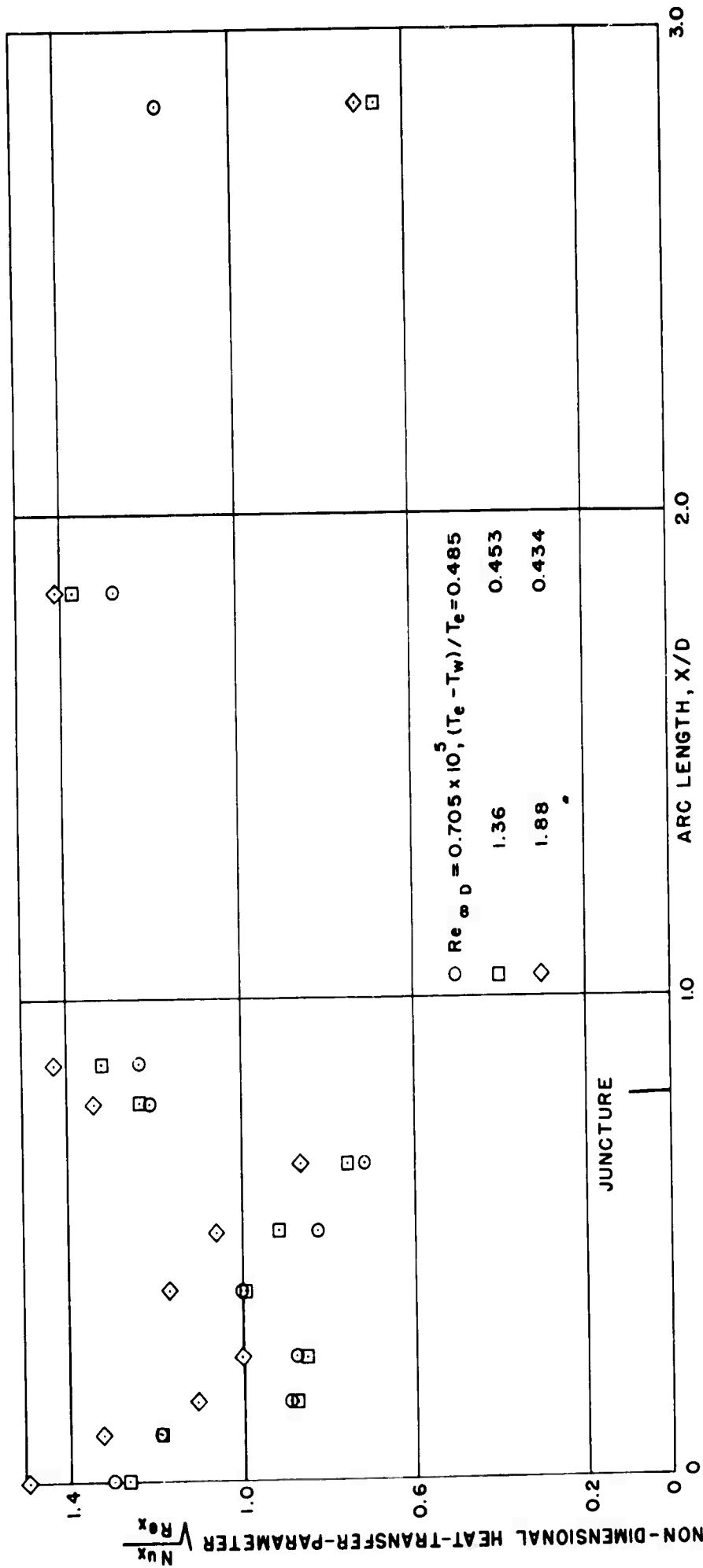


FIG. 8b NON-DIMENSIONAL HEAT-TRANSFER-PARAMETER DISTRIBUTION OVER
HEMISPHERE-CYLINDER

MACH NUMBER 5 AND $T_w / T_e \sim 0.517$

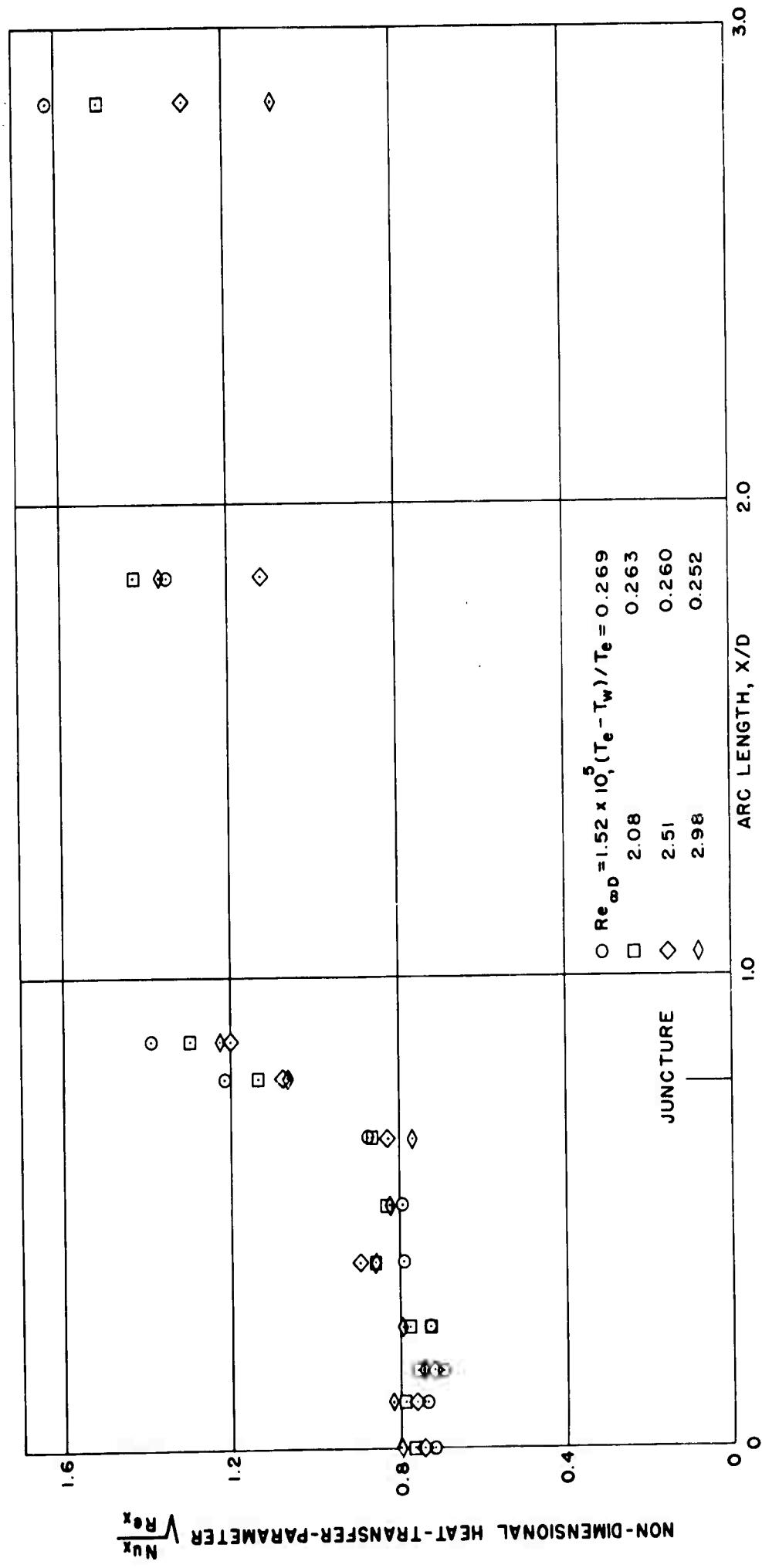


FIG. 8c NON - DIMENSIONAL HEAT - TRANSFER - PARAMETER DISTRIBUTION OVER HEMISPHERE - CYLINDER

MACH NUMBER 6.5 AND $\tau_w/\tau_e \sim 0.697$

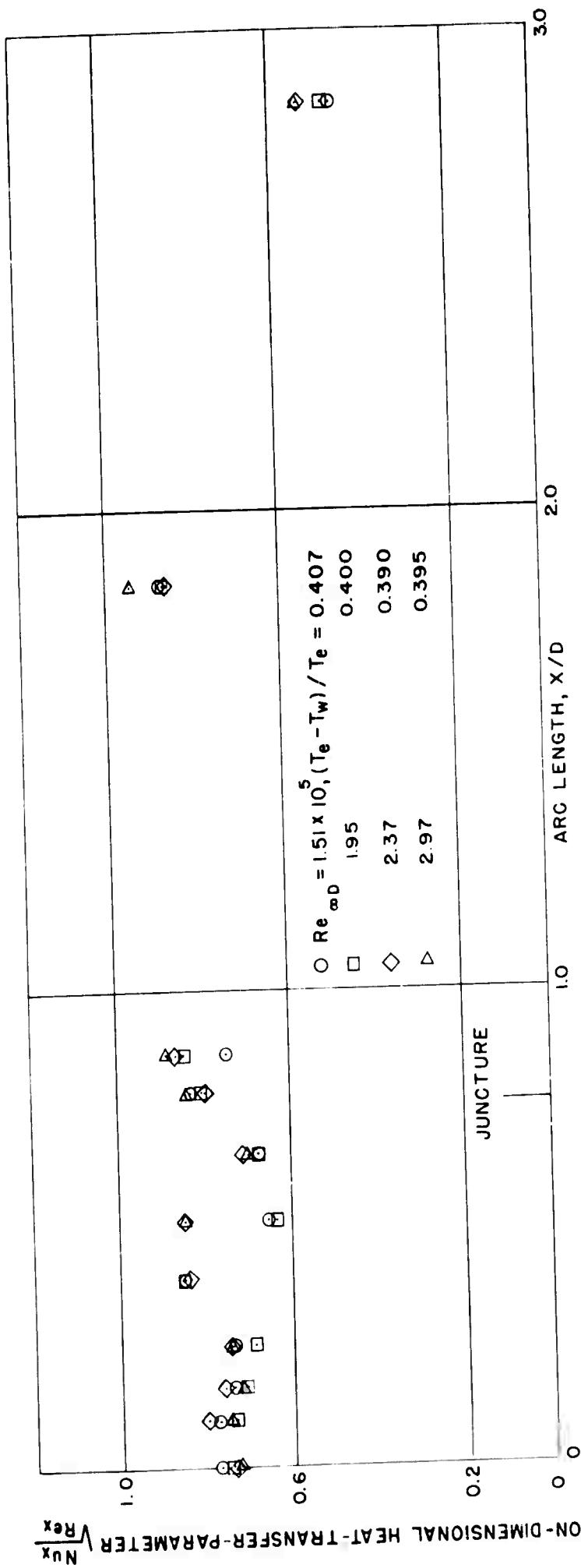


FIG. 8d NON-DIMENSIONAL HEAT-TRANSFER-PARAMETER DISTRIBUTION OVER
HEMISPHERE - CYLINDER

MACH NUMBER 6.5 AND $T_w / T_e \sim 0.565$

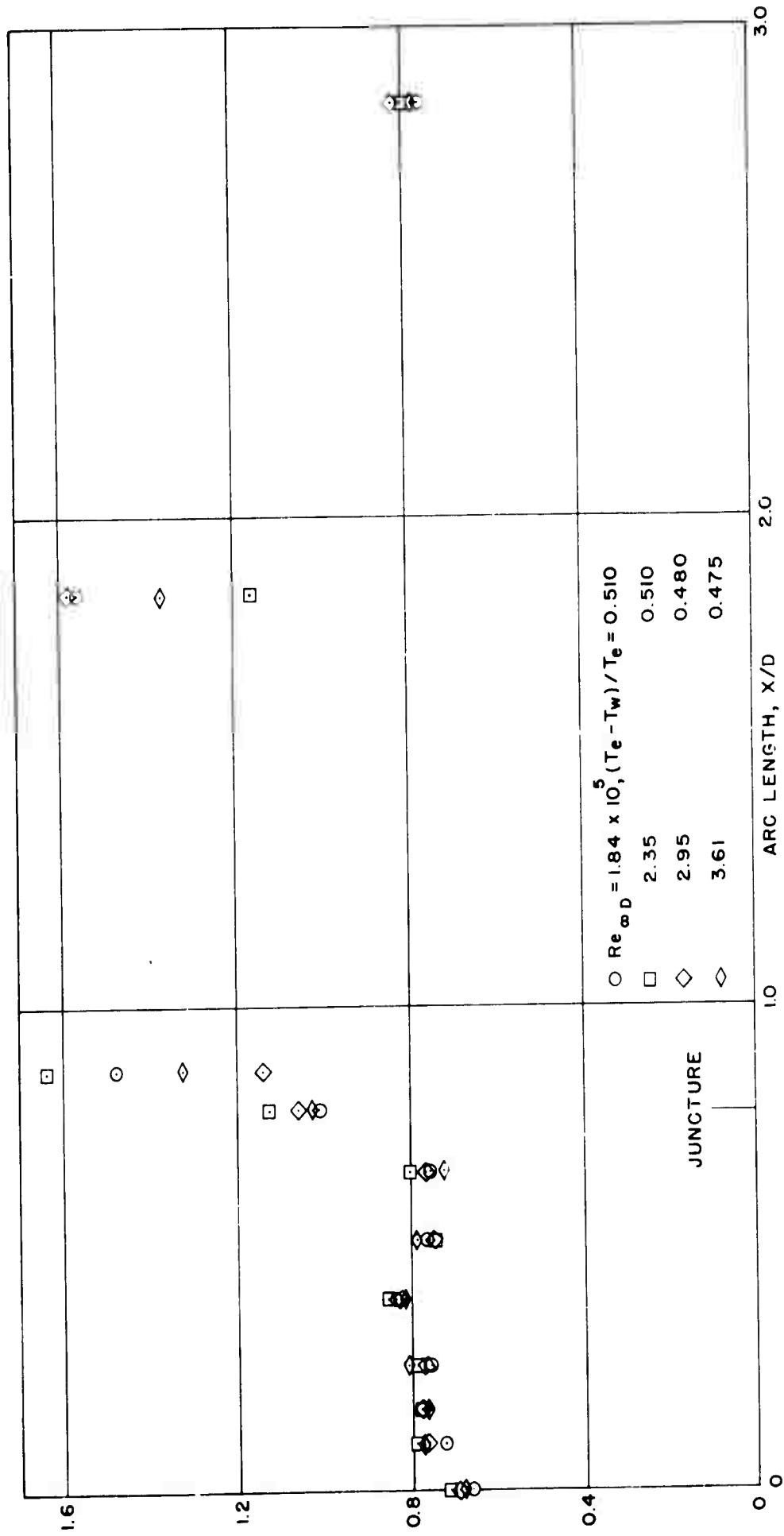


FIG. 8e NON - DIMENSIONAL HEAT - TRANSFER - PARAMETER DISTRIBUTION OVER HEMISPHERE - CYLINDER

MACH NUMBER 6.5 AND $T_w/T_e \sim 0.474$

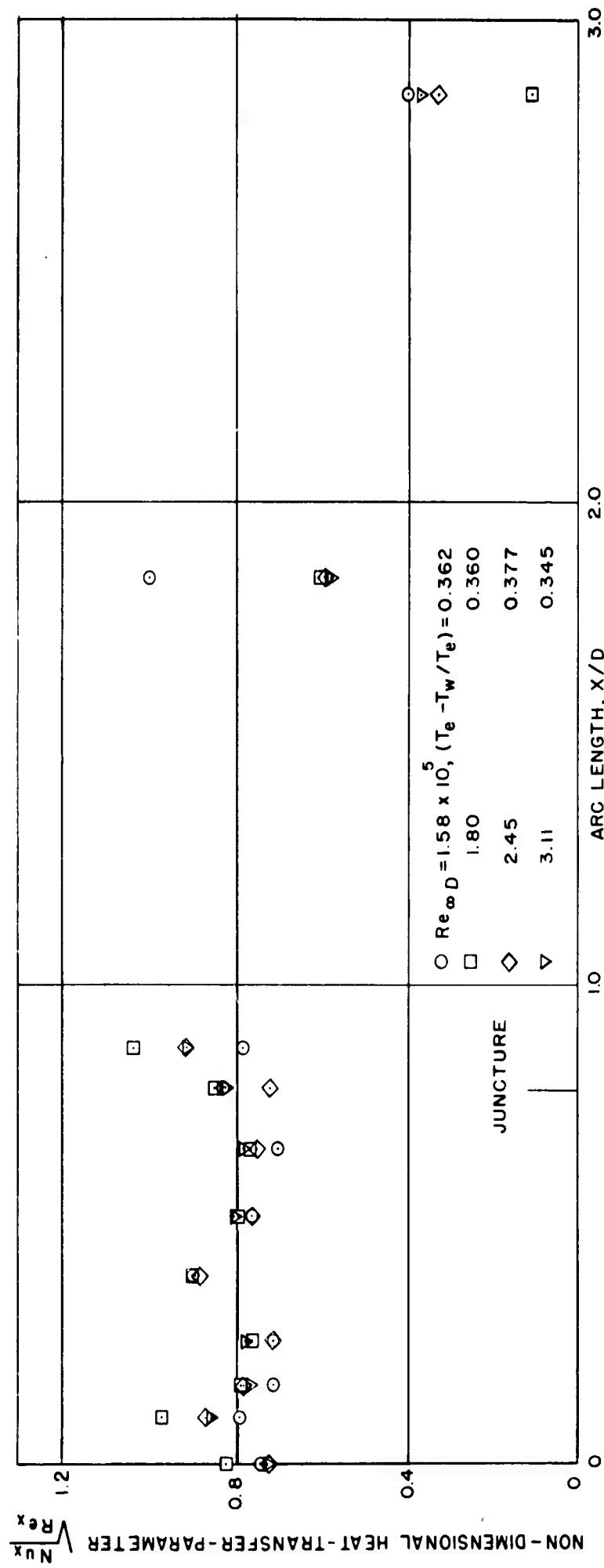


FIG. 8 f NON-DIMENSIONAL HEAT-TRANSFER-PARAMETER DISTRIBUTION OVER
HEMISPHERE-CYLINDER

MACH NUMBER 8 AND $T_w / T_e \sim 0.613$

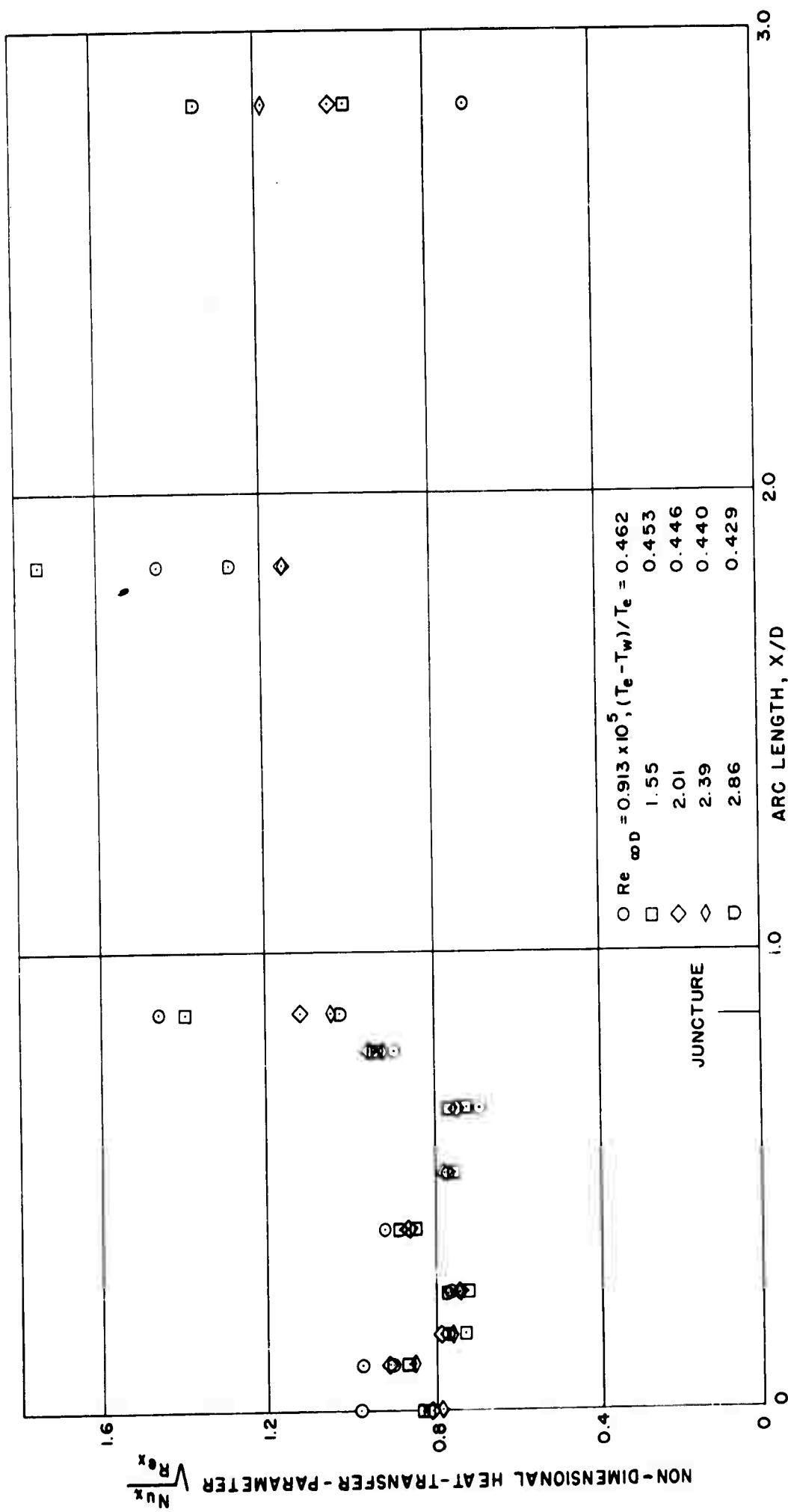


FIG. 8g NON-DIMENSIONAL HEAT-TRANSFER PARAMETER DISTRIBUTION OVER HEMISPHERE-CYLINDER

MACH NUMBER 8 AND $T_w/T_e \sim 0.522$

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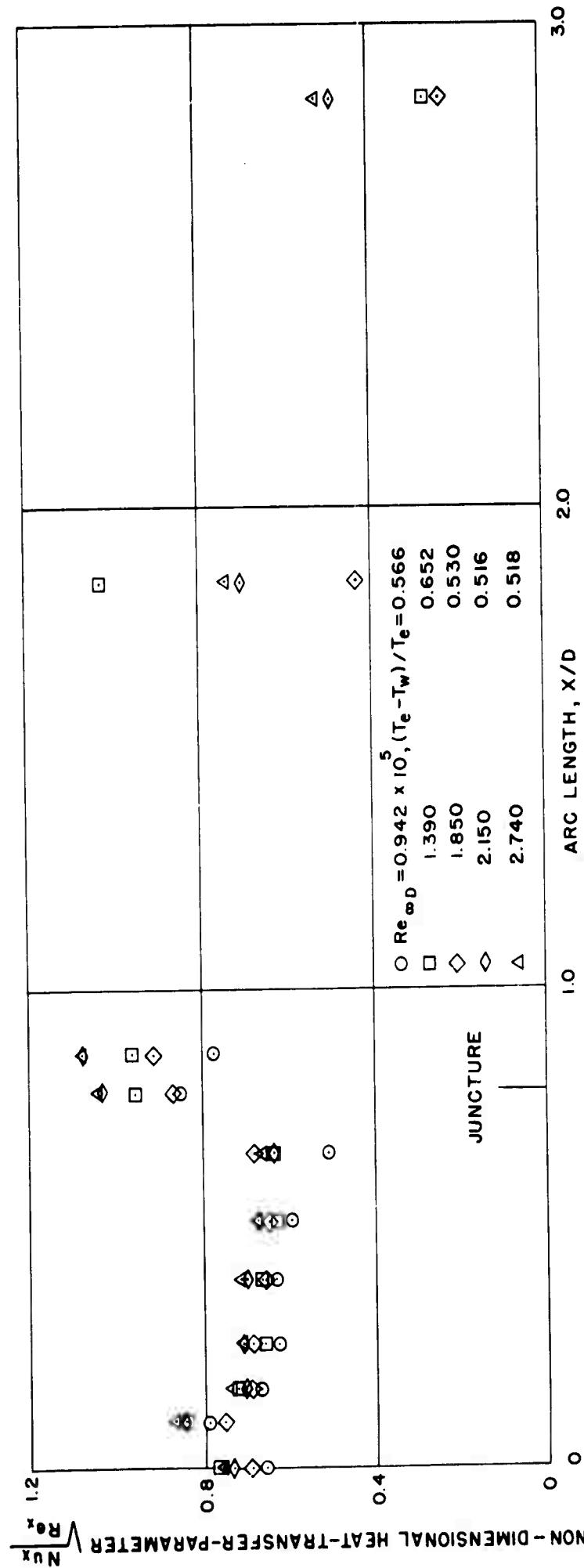


FIG. 8h NON-DIMENSIONAL HEAT-TRANSFER-PARAMETER DISTRIBUTION OVER HEMISPHERE-CYLINDER JUNCTURE

MACH NUMBER 8 AND $T_w/T_e \sim 0.440$

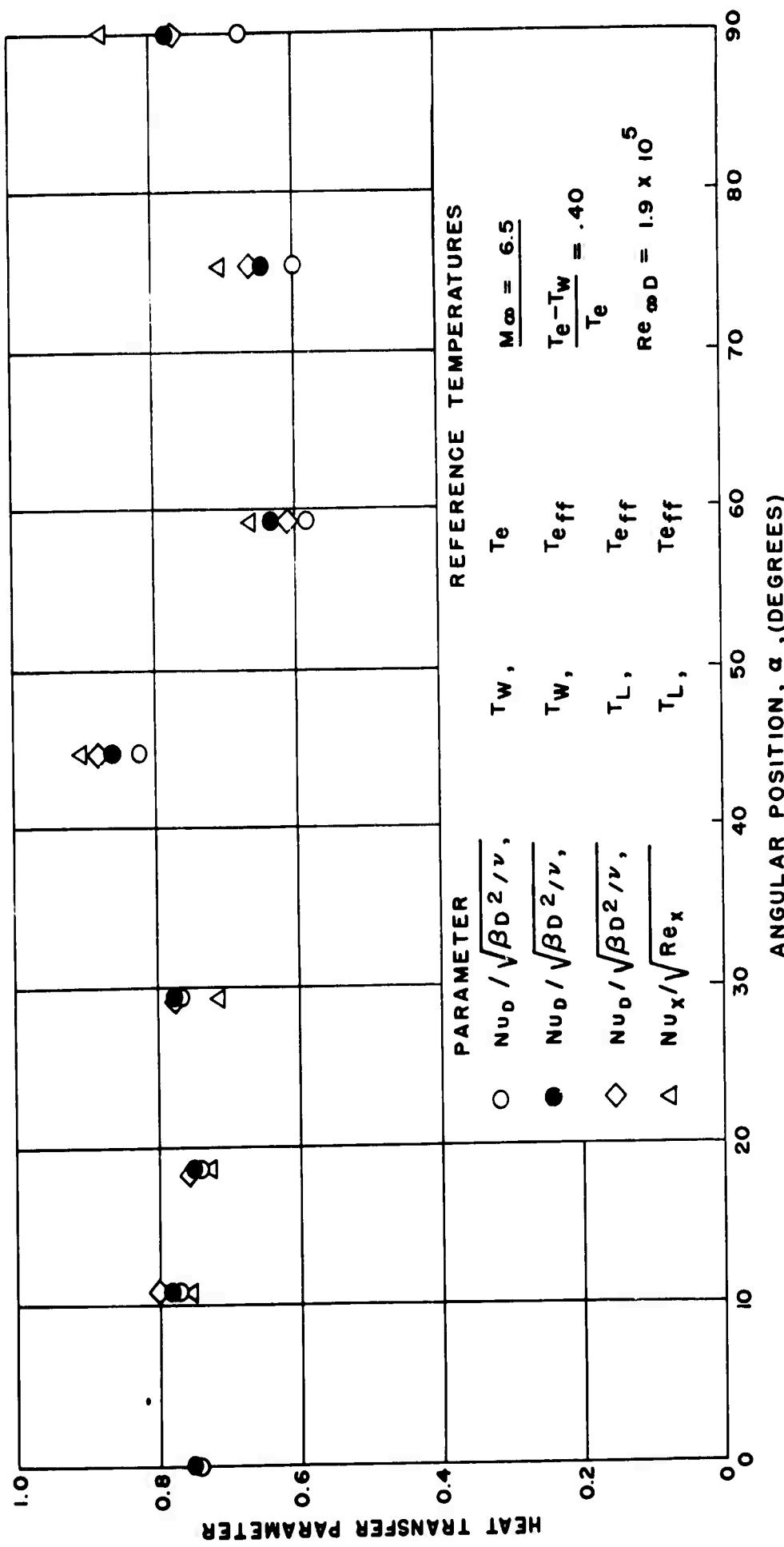


FIG. 9 COMPARISON OF HEAT-TRANSFER PARAMETERS
COMPUTED ON THE BASIS OF DIFFERENT REFERENCE VALUES

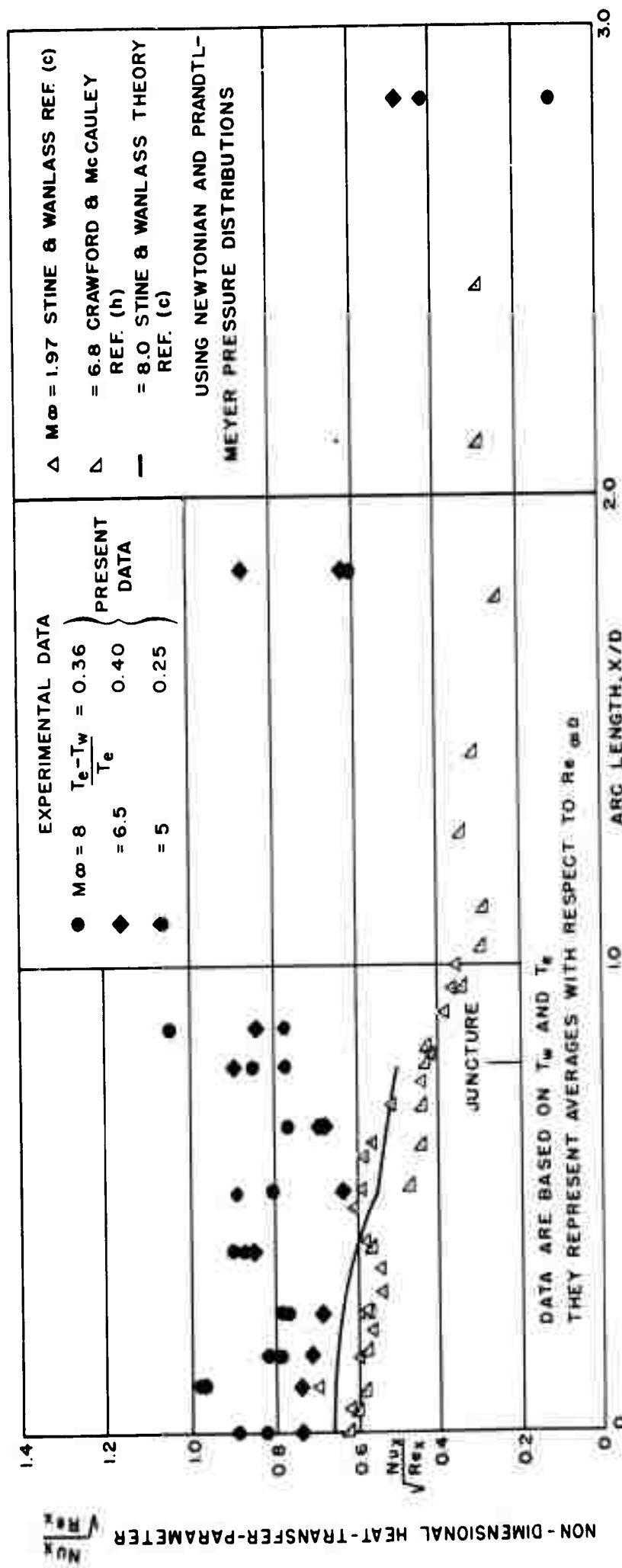


FIG. 10a COMPARISON OF PRESENT DATA WITH OTHER EXPERIMENTAL DATA AND WITH THEORY

$$\frac{Nu_x}{\sqrt{Re_x}} \text{ VS ARC LENGTH } x/d$$

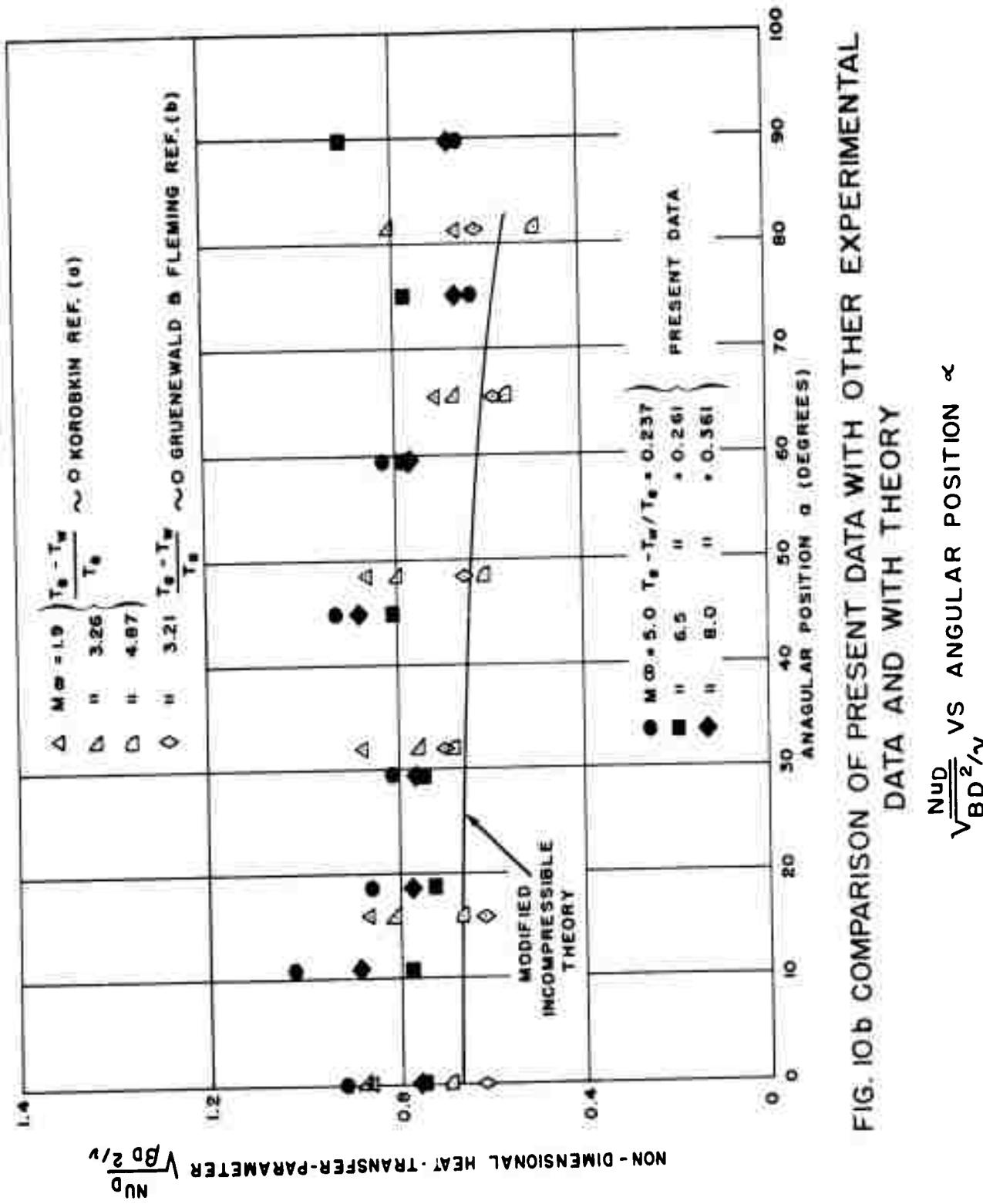


FIG. 10b COMPARISON OF PRESENT DATA WITH OTHER EXPERIMENTAL DATA AND WITH THEORY

$\frac{Nu_D}{\sqrt{BD^2/V}}$ VS ANGULAR POSITION α

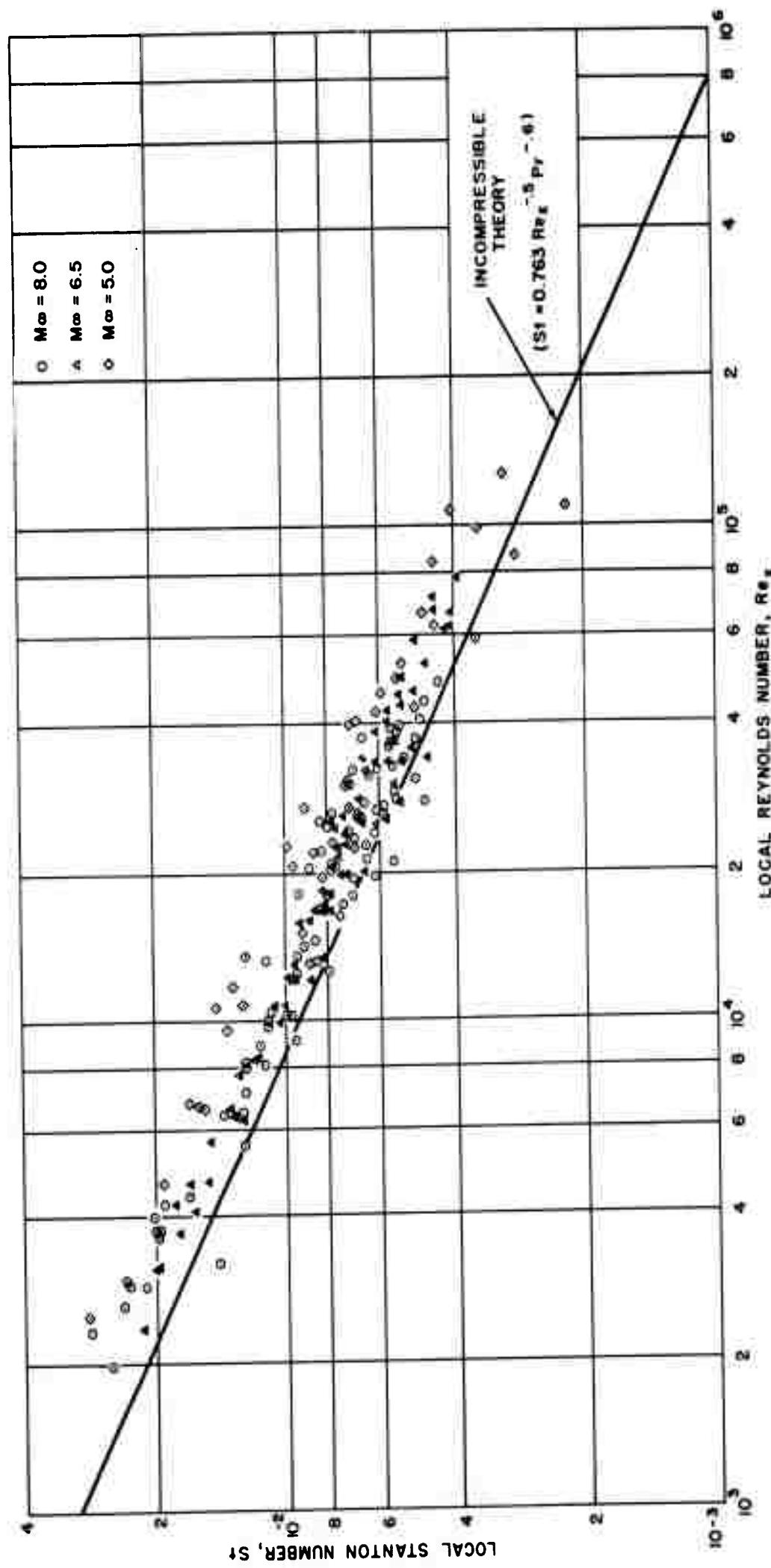


FIG. II VARIATION OF LOCAL STANTON NUMBER WITH LOCAL REYNOLDS NUMBER ALONG HEMISPHERE ($Pr = 0.70$)

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HEAT-TRANSFER CHARACTERISTICS OF A HEMISPHERE-CYLINDER AT HYPERSONIC
MACH NUMBERS, by E.M. Winkler and J.E. Danberg. 11 April 1957. 20p.
ILLUS., charts, tables, diagrs. (aeroballistic research report 336).
Projects NOL-133-1-50, and NOL-291.

UNCLASSIFIED

The heat-transfer characteristics of the laminar compressible boundary layer on a hemisphere-cylinder have been investigated at free-stream Mach numbers from 5 to 8, model wall to stagnation temperature ratio from 0.43 to 0.75, and Reynolds numbers based on body diameter from 70,000 to 700,000. Over the hemisphere, the local non-dimensional heat-transfer parameter, evaluated from the temperature differences measured across the model wall under steady-state conditions, was found to be approximately 20 percent larger than predicted for an isothermal body by Korobkin's modified incompressible theory.

1. Heat - Transference
 2. Boundary layer, compressible
 3. Boundary layer, laminar
 4. Cylinders - Boundary layer
 5. Cylinders - Heat transfer
 6. Mach number
 7. Reynolds number
- I. Title
II. Winkler, E.M.
III. Danberg, J.E., Jr. author
IV. Series
V. Series
VI. Project
VII. Project

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